



THERMODYNAMICS I

Q

Water

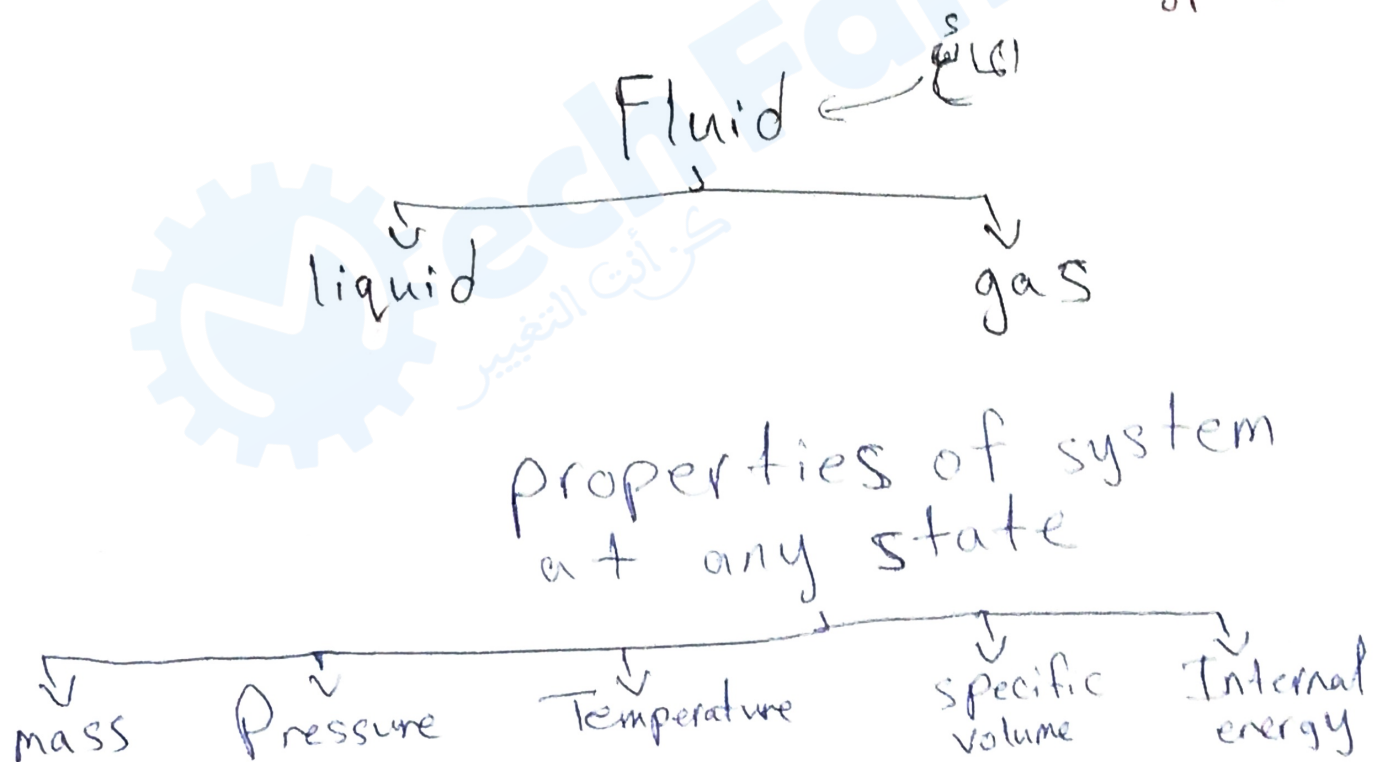
By :
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CHAPTER 1

INTRODUCTION AND BASIC CONCEPTS



* In thermodynamics, we study the fluid contained inside the system, we study its states, its properties in each state and the processes that happen to the fluid in their different types.



* We have also enthalpy and entropy but they will be discussed later.

Pressure $\equiv$ الضغط الذي يسيبه المائع على جدران النظام
(KPa)

Internal energy (u) $\equiv$ الطاقة الداخلية لجزيئات المائع داخل النظام
(KJ)

Specific Volume (v) $\equiv$ حجم الـ 1kg من المائع داخل النظام
(m³/kg)

مثال توضيحي لـ Specific Volume

$$v = \frac{V}{m}$$

$\uparrow$ m³/kg $\uparrow$ m³ $\uparrow$ kg
 Water
 $v = 5 \text{ m}^3/\text{kg}$ $\leftarrow v = 5 \text{ m}^3/\text{kg}$

* يعني لو لدينا 1kg من الماء من هذا النظام سيكون عبارة عن 5m³

Specific energy $\equiv$ طاقة الـ 1kg من المادة (او المائع) داخل النظام
(KJ/kg)

مثال توضيحي :-

Water
10 KJ/kg

* يعني كل 1kg من الماء قوي طاقة مقدارها 10KJ

Density $\equiv$
(kg/m³)

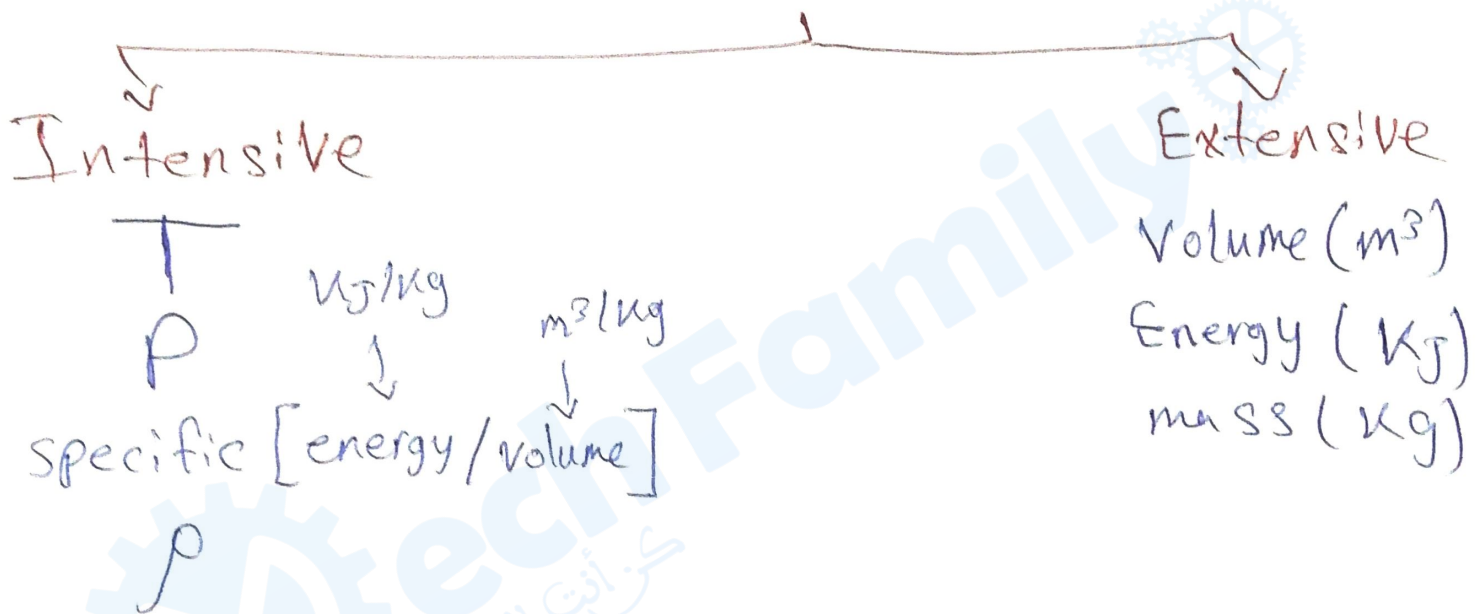
كتلة الـ 1 m³ من المادة
او المائع داخل النظام

$$\text{density} \rightarrow \rho = \frac{1}{V} \quad \text{specific volume}$$

$$\rho = \frac{m}{V}$$

↑ kg/m³ ↑ m³

* All these properties can be classified as:-



طريقة هامة :-

لا يمكن الجمع بين الـ intensive properties (بعضها يمكن في حالة الـ extensive)

توضيح :-

نفترض ان لدينا نظامين 1 و 2

$$v = 5 \text{ m}^3/\text{kg}$$

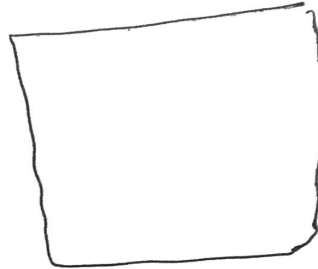
والاخر

$$v = 10 \text{ m}^3/\text{kg}$$

$$v = 5 \text{ m}^3/\text{kg}$$

$$v = 10 \text{ m}^3/\text{kg}$$

اِردنا جمعها في نظام جديد
(تقريبها)

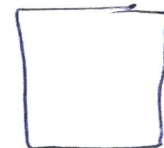
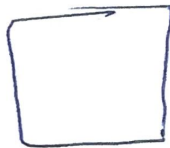


هل يكون الـ v للنظام الجديد $15 \text{ m}^3/\text{kg}$ ؟
لا جايه Δ

وهذا ينطبق على جميع intensive properties
بمعنى لا يمكن جمع صفتين او درجتين
حراره

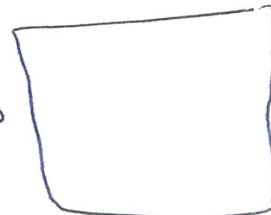
ولكن هذا ممكن في حالة الـ extensive

7 kJ
4 kg
5 m³



5 kJ
10 kg
3 m³

12 kJ
14 kg
8 m³



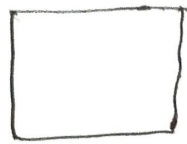
Types of systems

closed system
(control mass)

* No mass exchange

Ex:-

rigid tank $\Rightarrow$



piston cylinder $\Rightarrow$



open system
(control volume)

* There is mass exchange

Ex:-

- Turbine
- compressor
- pump
- Nozzle

* Difference between isolated and insulated systems:-

Isolated:-
system

نظام مغزل عن المحيط
لا يتبادل الطاقة أو الكتلة
مع النظام المحيط

Insulated:-
system

نظام مغزل حرارياً
(No heat exchange)

$$Q_{in} = Q_{out} = 0$$

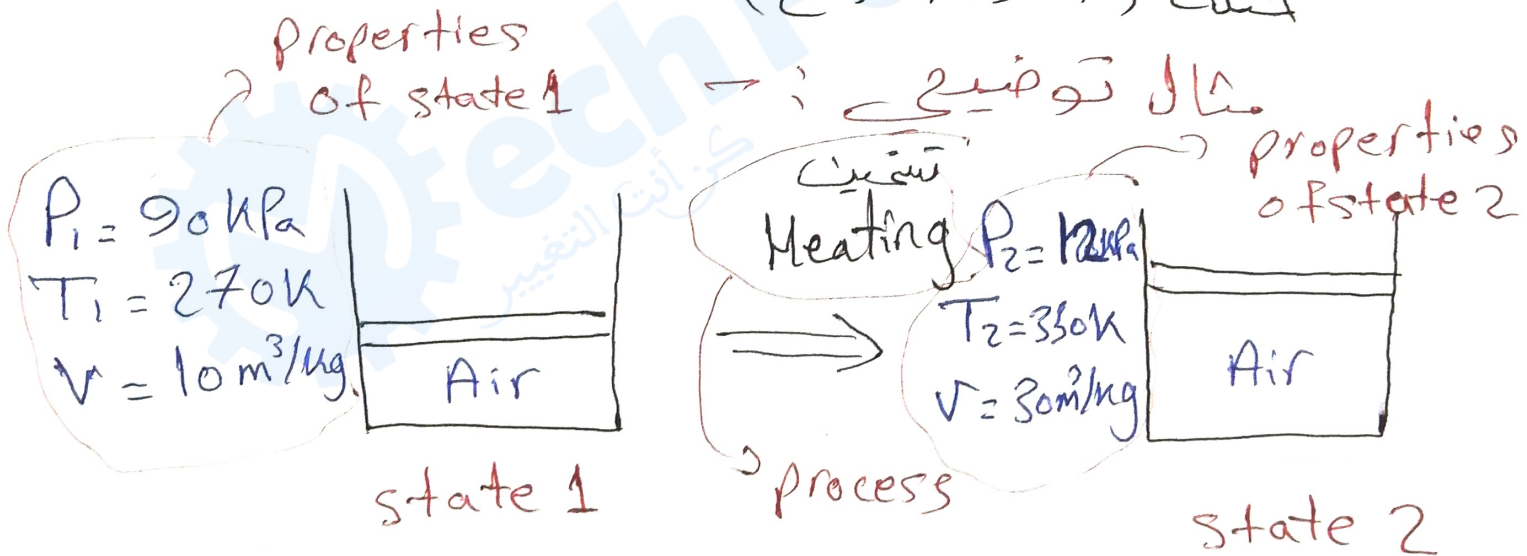
قد يتبادل الكتلة وأنواع الطاقة الأخرى
مع النظام المحيط.

* النظام (system) في أي لحظة يكون في حالة (state) معينة ، وفي هذه الحالة تحمل النظام خصائص (properties) معينة

$$(P, T, v, h, s)$$

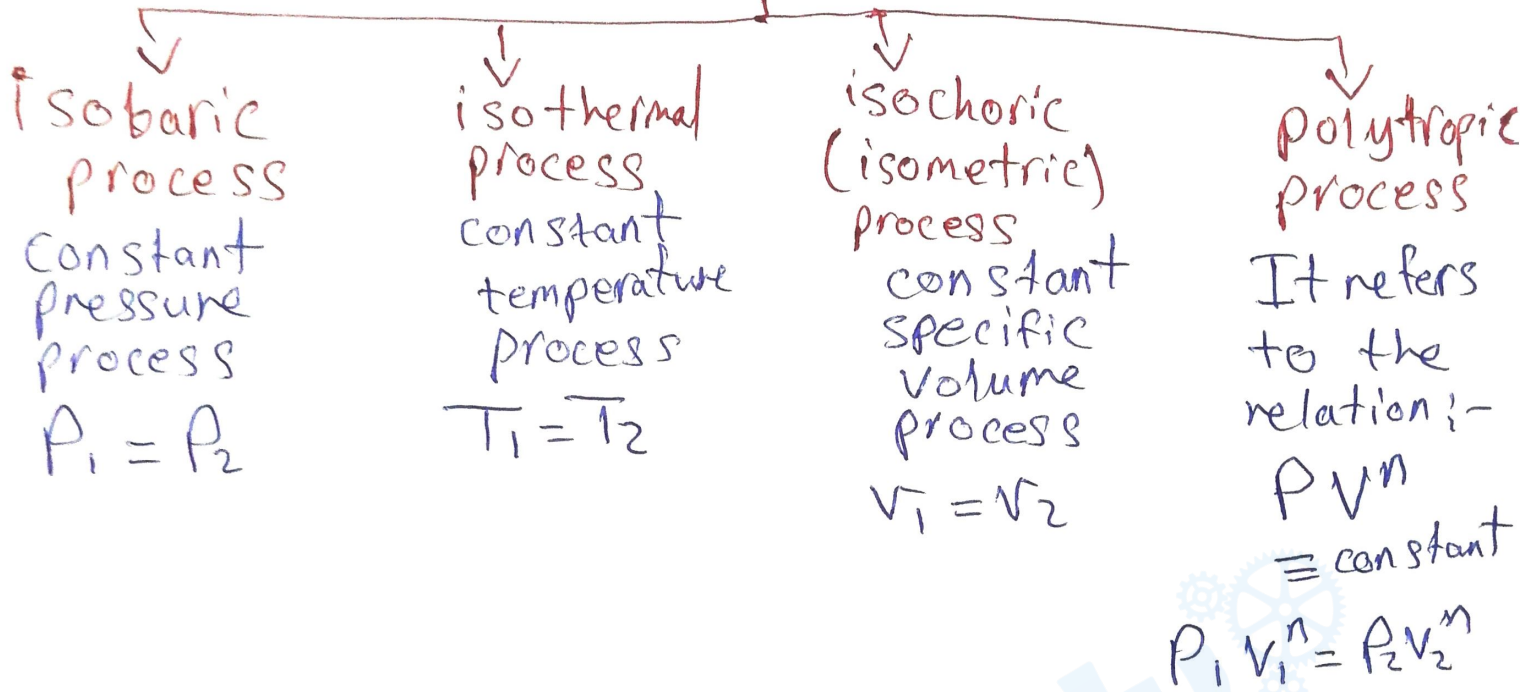
حتى تحدث عملية (process) وتنقل هذا النظام من هذه الحالة إلى حالة أخرى بخصائص جديدة

هذه العملية (process) يمكن أن تتضمن تسخين ، تبريد (مروحة / مدفئة) ، انتقال كتلة (دخول / خروج) ...

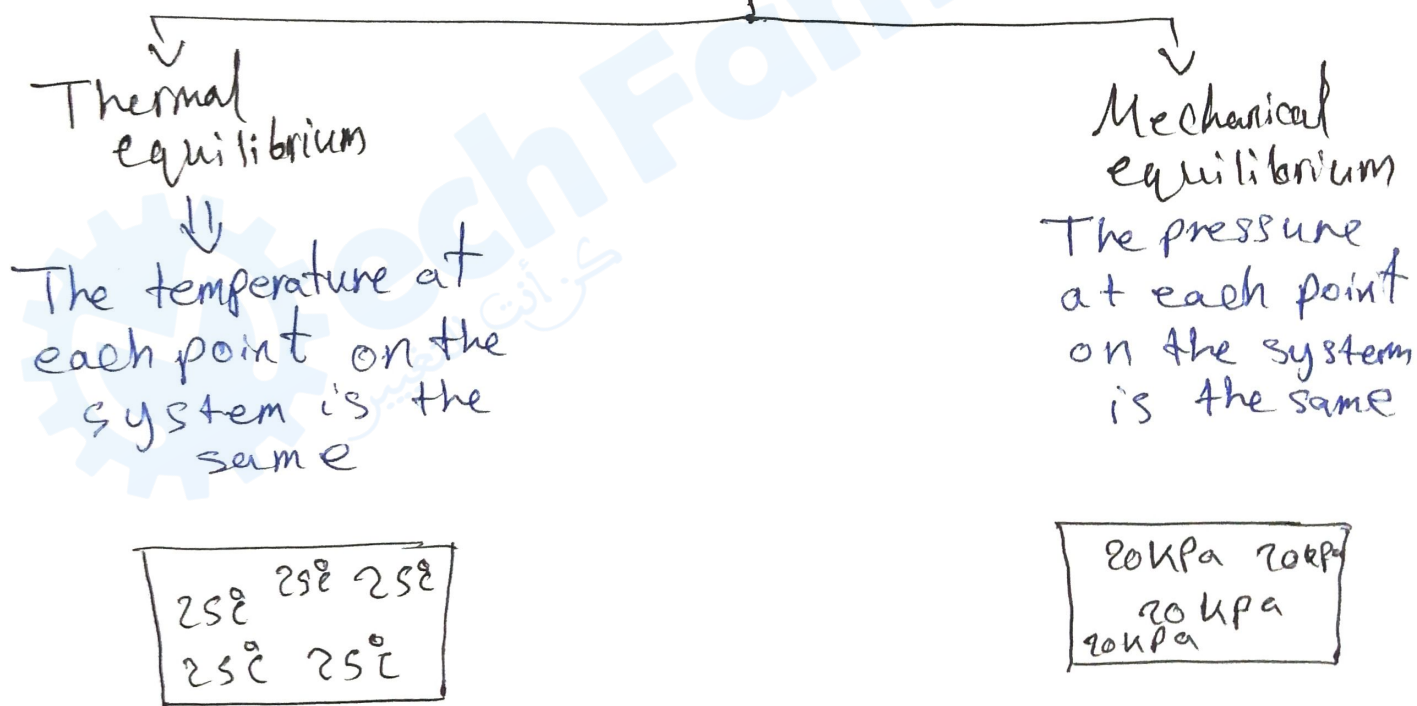


* الأرقام غير دقيقة ، فقط للتوضيح

Thermodynamic processes



Types of equilibrium



وفي مادة الترموديناميك على اي حال (state) يكون

النظام في حالة Thermal + Mechanical equilibrium

يعني اذا قال لك في المسألة ان $P_1 = 100\text{kPa}$:

يعني جميع النقاط لها نفس الضغط والقياس هو 100kPa
 وكذلك الامر بالنسبة لدرجة الحرارة على هذه الحالة

Pressure :-

Normal force exerted by a fluid per unit area.

$$1 \text{ bar} = 100 \text{ kPa}$$

$$1 \text{ atm} = 101.325 \text{ kPa}$$

$$1 \text{ MPa} = 1000 \text{ kPa}$$

- Atmospheric pressure (P_{atm}):-

ضغط الجو الكلي و يقاس به
Barometer

- Absolute pressure (P_{abs}):-

ضغط مطلق

- Gage pressure (P_{gage}):-

الفرق بين P_{atm} و P_{abs}

$$P_{gage} = P_{abs} - P_{atm}$$

وهناك جهاز يسمى (Pressure gage) يقيس

P_{gage} في tank

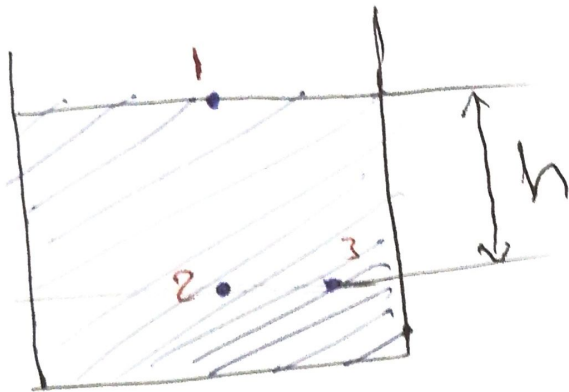
- Vacuum pressure (P_{vac})

فرق الضغط بين

$$P_{atm} > P_{abs}$$

$$P_{vac} = P_{atm} - P_{abs}$$

pressure + depth :-



* الضغط على سطح
الساكن (على نقطة من
الوسط) P_{atm}

$$\text{so } P_1 = P_{atm}$$

* النقاط العميقة والعميقة ضغطها أعلى لنفس المائع
وذلك لأن المائع يتركز أكثر في الطبقات العميقة

$$\text{so } P_2 > P_1$$

* الضغط يمتد على الارتفاع (ضغط المائع)
وبالتالي النقاط التي على نفس الارتفاع (على نفس
الخط الأفقي) يتساوى ضغطها

$$\text{so } P_2 = P_3$$

فارق الضغط بين أي نقطتين عملاً
بينهما مائع في حالة سكون :-

$$\Delta P = \rho g h$$

الارتفاع بين النقطتين $\rightarrow$
كثافة السائل الذي يعمل بين النقطتين $\downarrow$
9.81 $\downarrow$

$$\rho :- \text{Kg/m}^3$$

$$g = 9.81 \text{ m/s}^2$$

$$h :- \text{m}$$

$$\Rightarrow \Delta P :- P_a$$

So:-

$$P_{2 \text{ or } 3} - P_1 = \rho g h$$

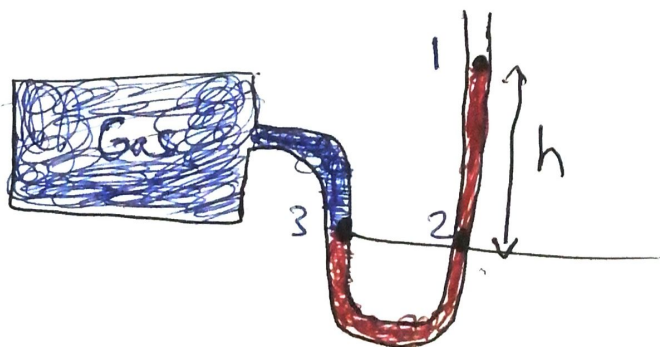
so the general equation is:-

$$P_{\text{below}} = P_{\text{above}} + \rho g h$$

Manometer:-

It's a device that measures the pressure in a tank

*It consists of U tube, which is made of plastic which contains one or more heavy fluid like oil, water, mercury



$$P_2 = P_3$$

$$P_{2 \text{ or } 3} = P_1 + \rho g h$$

$$\begin{array}{l} \rightarrow SG = \frac{\rho_{\text{fluid}}}{\rho_{\text{water}}} \\ \text{specific gravity} \end{array}$$

$$\rho_{\text{water}} = 1000 \text{ kg/m}^3$$

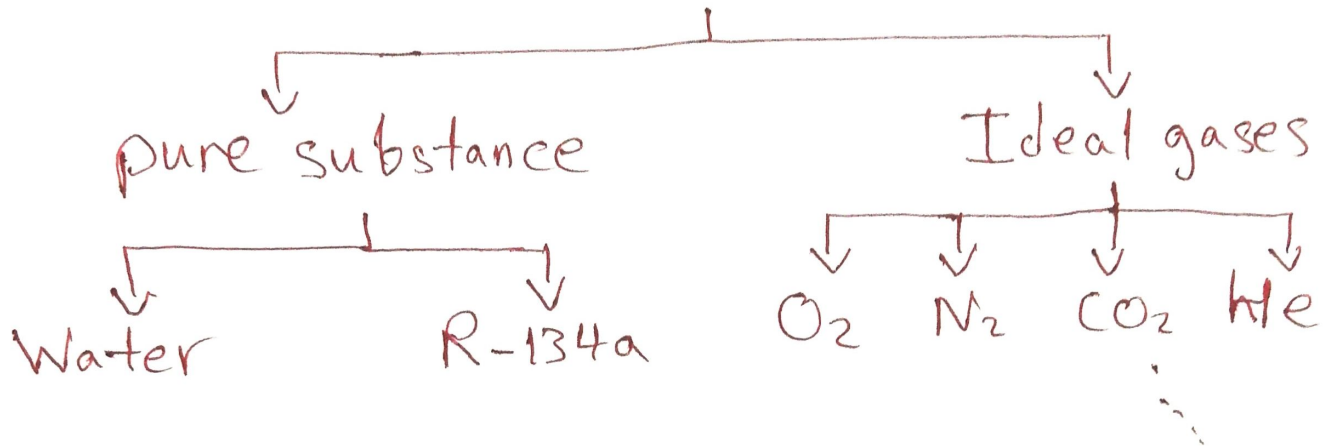
End of chapter 1



CHAPTER 3

PROPERTIES OF PURE SUBSTANCES

Substances that we deal with in thermodynamics



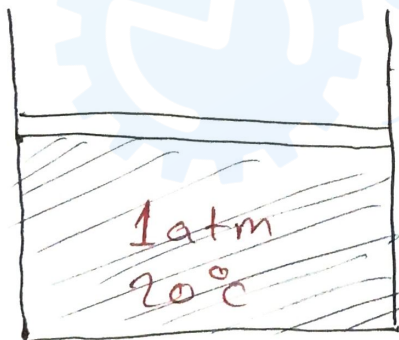
pure substances
behavior! -

سلوك المواد النقية! -

* لنفترض أن لدينا سائداً داخل
piston cylinder تحت ظروف! -

$$P = 1 \text{ atm}, T = 20^\circ\text{C}$$

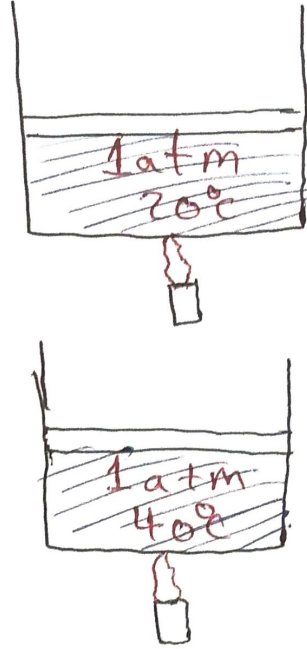
* تحت هذه الظروف، يكون الماء بحالة
سائداً! -



Subcooled liquid
or
compressed liquid

و يجب التركيز على هذه المسميات جيداً

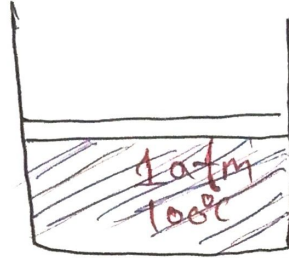
* ثم وضعنا شعاع (Heat source) أسفل ال cylinder



* سوف ترتفع درجة الحرارة بشكل تدريجي
، لنقل الماء وصلت إلى 40°C ، ويبقى الضغط ثابتاً
، حيث أن الضغط يعتمد على وزن المكبس بالإضافة
إلى الضغط الخارجي (الضغط الجوي P_{atm})
، وفي هذه العملية ، يكون المكبس حراً ، فيبقى
الضغط ثابتاً ، وظلال ارتفاع درجة الحرارة التدريجي
، يرتفع المكبس ولكن بشكل بسيط ، حيث أن جزيئات
الماء تضغط على المكبس خلال حركتها الناتجة عن
الطاقة الحركية المضافة لها نتيجة التسخين

* تحت هذه الظروف التي وصلنا لها (1 atm , 40°C)
يبقى الماء في حالة -
compressed or subcooled liquid

* خلال استمرارنا في التسخين (Heat addition)، تستمر درجة الحرارة في الارتفاع التدريجي ويبقى الضغط ثابتاً، حتى نصل إلى $(1 \text{ atm}, 100^\circ\text{C})$



* عند هذه الظروف، نصل إلى حالة تسمى
saturated liquid

* saturated liquid: - It's a liquid that is about to vaporize

* حيث أن أي مقدار إضافي من الطاقة الحرارية (Q) سيؤدي إلى بدء تبخير هذا السائل من الماء

* نستمر في إضافة الطاقة الحرارية التي مصورها الشعاع، وهنا يبدأ السائل بالتبخير تدريجياً

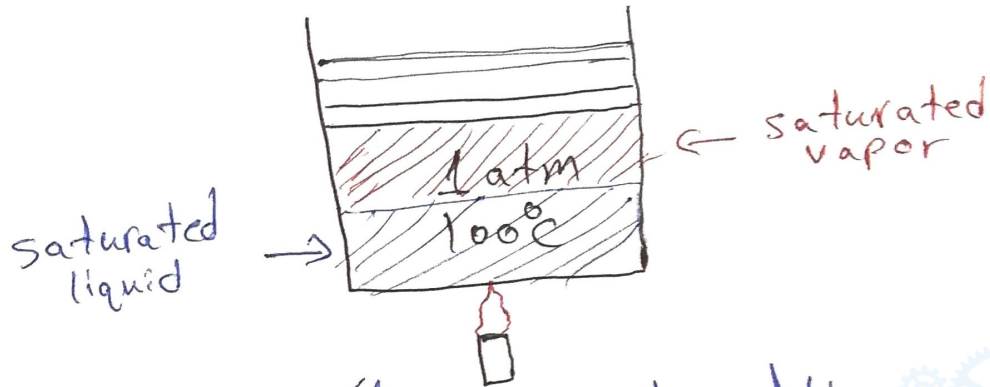
* وخلال تحول السائل التدريجي إلى بخار، تبقى درجة الحرارة ثابتة، كما ذكّر

* لأن الطاقة الحرارية المضافة (Heat) تستعمل على تفكيك الروابط بين جزيئات السائل لتحويله إلى بخار، فلا يكون لهذه الطاقة القدرة على رفع درجة

الحرارة

* وخلال التحول التدريجي من السائل إلى البخار ، نحن نكون في حالة -

saturated liquid vapor mixture

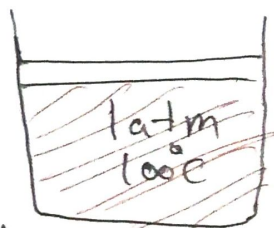


* وخلال هذا التحول أيضاً ، يرتفع المكبس بشكل كبير ليسبب وجود Vapor بالداخل ، حيث أنه من المعروف أن الطاقة الحركية للبخار أو الغاز بشكل عام أكبر من السائل ، وبالتالي مع زيادة كمية البخار يزداد دفع المكبس لأعلى .

* تستمر هذه العملية حتى يتحول كل السائل إلى بخار (100% vapor) ، ويكون هنا المائع في حالة

* ارتفاع
المكبس $\equiv$ زيادة
الـ specific
volume

saturated vapor

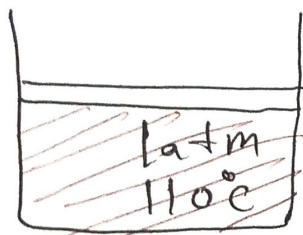


* saturated vapor :- A vapor that is about to condense

* بحيث أن أي تبريد (سحب للحرارة) من البخار سيؤدي مباشرة إلى تكثفه (تحوله إلى سائل) تدريجياً

* ومع استمرار تسخين (إضافة طاقة حرارية) لهذا الـ **saturated vapor** ، سوف تستأنف درجة الحرارة في ارتفاعها ، وطبيعياً يبقى الضغط ثابتاً وذلك السبب قبل ذلك ، وسوف يرتفع المكثف بشكل أكبر وأكبر .

* وبإرتفاع درجة حراره الـ **sat. vapor** ، يصل البخار (المائع) إلى حالة تسخين :-
superheated vapor

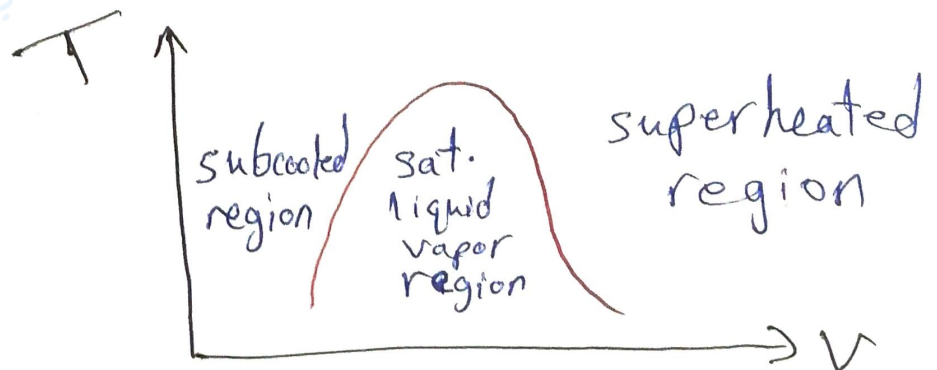


ظروف يكون
عندها المائع
superheated vapor.

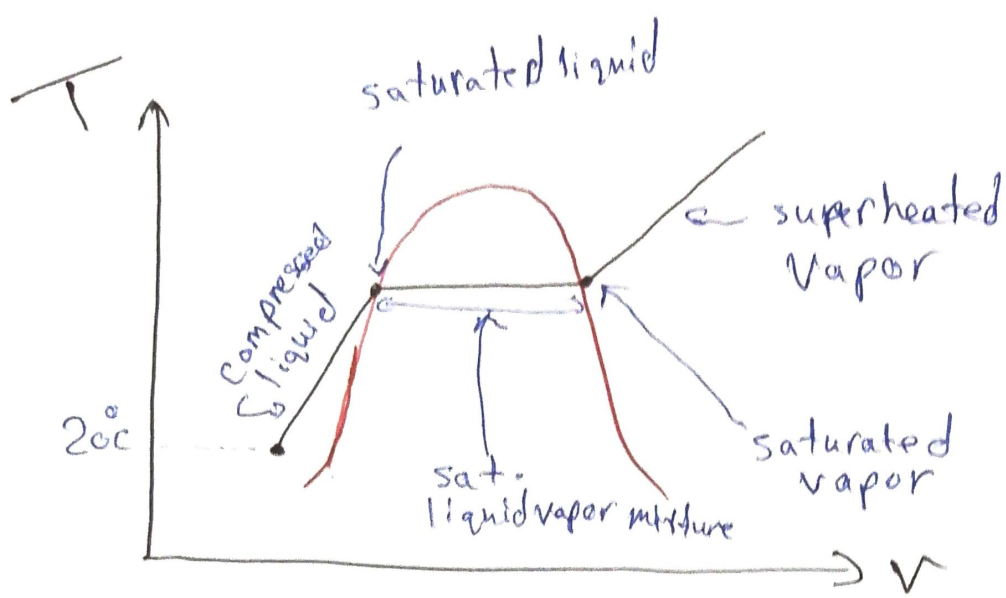
* هذه العمليات وهذا الـ **procedure** ينطبق على الـ **pure substances**

T-v
diagram

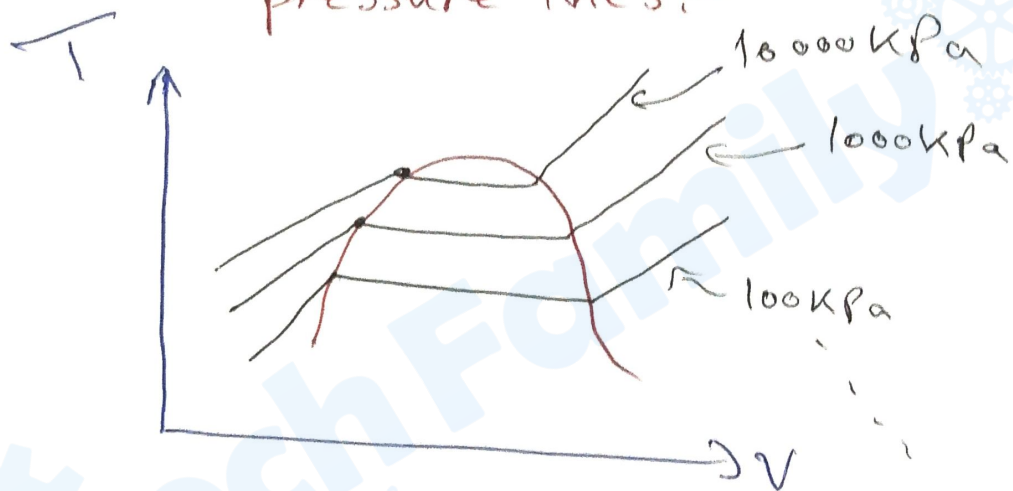
* ويمكن تمثيل ذلك على



Temperature - specific volume
diagram
with regions.

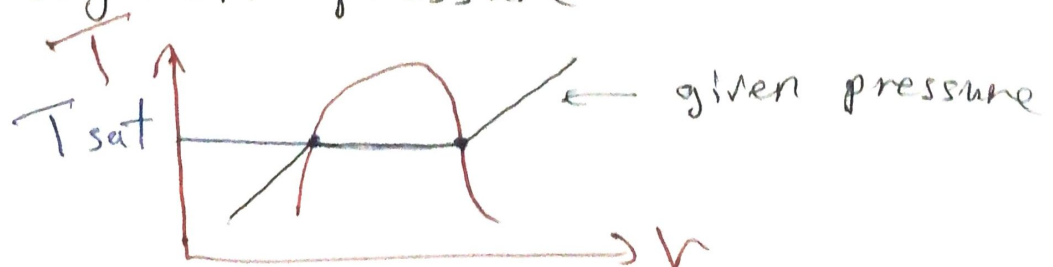


* In $T-v$ diagram, constant pressure lines:-



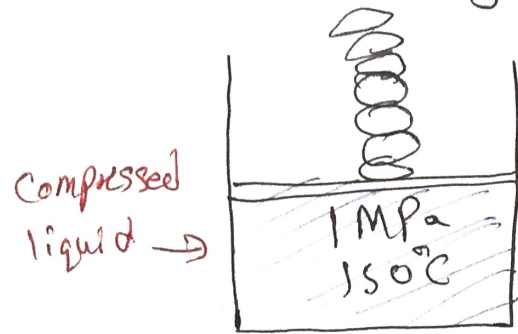
Saturation temperature
+
saturation pressure
 $P_{sat} + T_{sat}$.

saturation temperature:- It's a temperature at which a pure substance changes phase at a given pressure



تجربة اخرى توضح تأثير وسلوك الضغط:

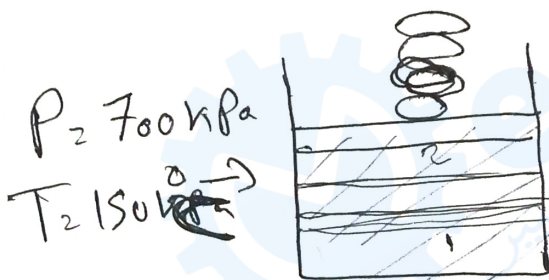
لفرضنا ان لدينا ماء داخل $Piston$ cylinder على



ظروف $(\frac{P_1}{T_1} = \frac{P_2}{T_2})$ ، وطبعا
فوقا للمكبس توجد مجموعة
من الاجزاء التي هي مصدر
هذا الضغط

حيث اننا كما نعلم، ضغط المائع في $Piston$ cylinder
يعتمد على وزن المكبس والضغط الجوي، والاعمال
هنا تزييد من الوزن الذي تحمله المكبس

ثم به اننا بزيادة الاجزاء بالتدريج



ما سيحدث ان الضغط على
المائع سيقول ، وسيبدأ المكبس
بالارتفاع لكن بشكل بسيط
جدا

وخلال هذه العملية ستثبت درجة الحرارة

وخلال الاستمرار ما ينخفض الضغط تدريجيا وتبقى

الماء على حاله $compressed$ liquid حتى يصل الضغط

الى 476.16 kPa ، فكون قد وصلنا الى مرحلة الاشباع

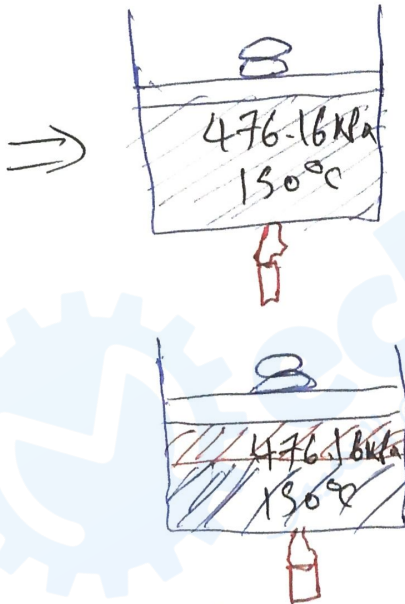
(saturated liquid)

ثم بعد ذلك نضع شحنة أسفل النظام لنمد الماء بالـ *heat*

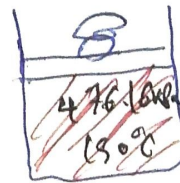
وظائره هذه الهيئت ، تحويل السائل الى بخار بالتدريج
دون احداث اي تغيير في درجه الحرارة او الضغط
وظلال هذه العملية :-

يتحول السائل الى بخار تدريجيا
وتبقى P و T ثابتتان
والمكبس يرتفع اكثر بسبب به ايه
تكون البخار

sat.
liquid



ويستمر ذلك حتى
تنتهي عملية التحول
فيصبح السائل كله
بخار



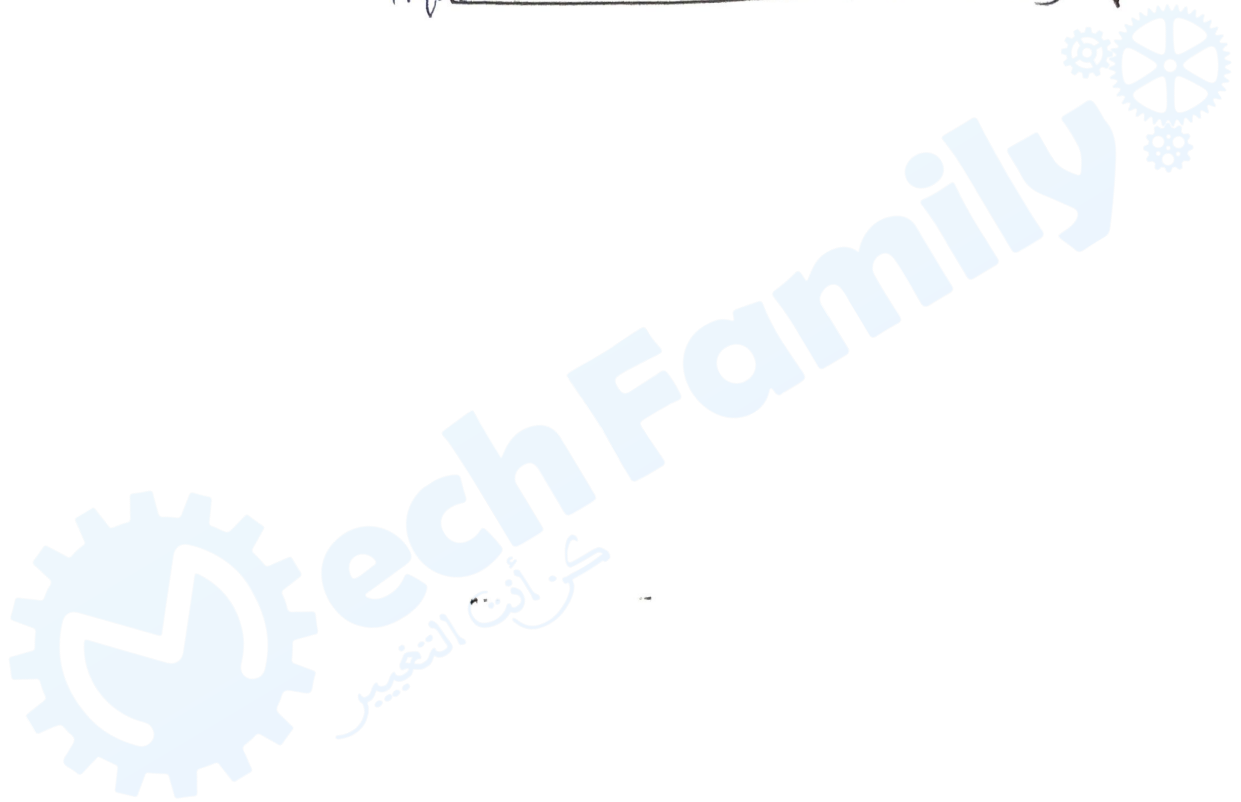
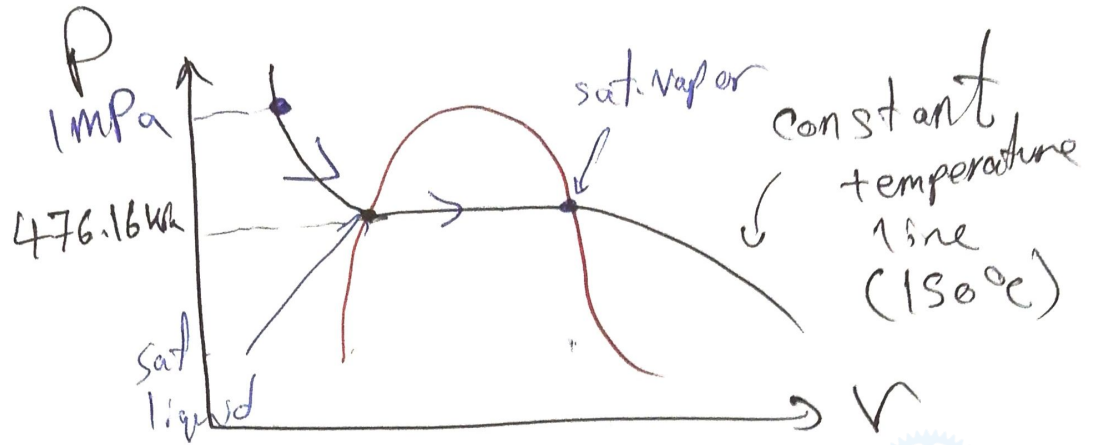
بعد التحول الى بخار ، تكون قد وصلنا الى حالة
saturated vapor ، ثم طبعا نزيل الشحنة

وبعد استئناف ازالة الاحجار ، يستمر الضغط في الهبوط
، حيث انه بمجرد ازالة اي مقدار من الوزن بعد

التحول الى *Sat. Vapor* ، تكون قد دخلنا في حالة الـ

super heated vapor

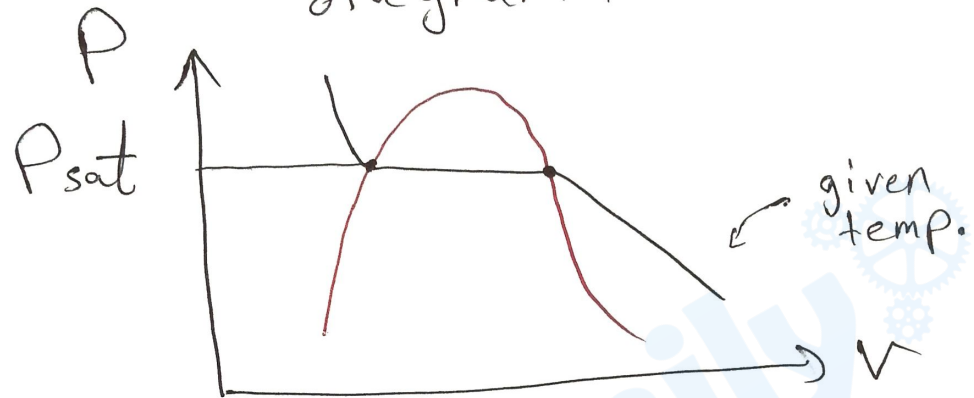
لوہ والا سیلڈر = اسٹیل
 پستون والا سیلڈر = پستون
 پستون والا سیلڈر = پستون
 پستون والا سیلڈر = پستون



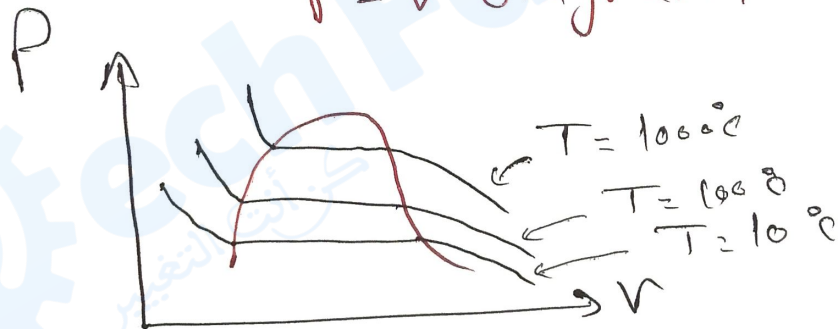
Saturation pressure :-

It's the pressure at which a pure substance changes phase at a given temperature

Pressure - Specific volume diagram :-

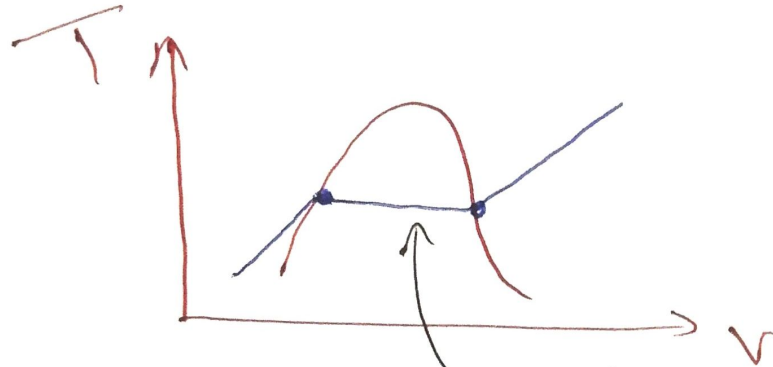


constant temp. lines on P-v diagram :-



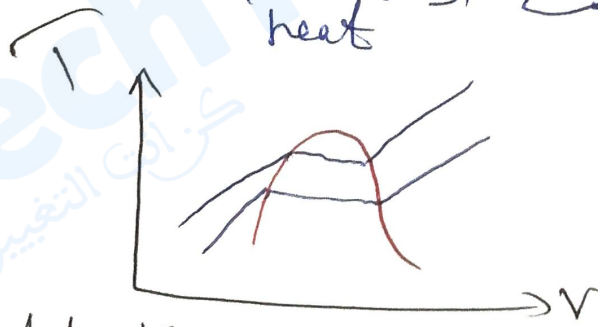
sensible heat :- It's the heat that causes a change in the temperature

latent heat :- It's the heat that doesn't cause any change in the temperature, but it causes a change in the phase



(sat. vapor الى sat. liquid للتحويل) latent heat

* كلما زاد الضغط، زادت T_{sat} وبالتالي قلت latent heat



* كلما زاد الضغط، قل طول المنطقة، قل T_{sat} و P

وينطبق أيضاً هذا الكلام على $P-v$ diagram

$T \uparrow \uparrow$

$P_{sat} \uparrow \uparrow$

latent heat $\downarrow \downarrow$

Tables property:-

* أي مائع في أي حاله يكون له خاصائص معينة
(لكل حاله خاصائص)

* state postulate

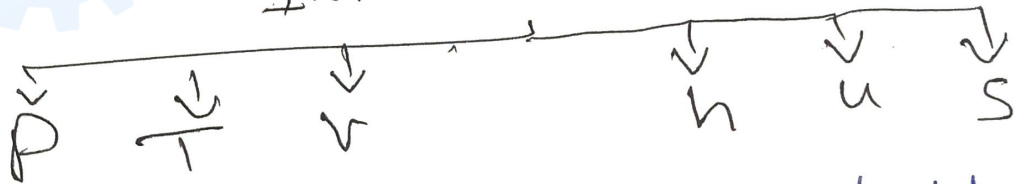
-: (ثمة خاصائص الحاله)

* The principle of state postulation says:-

any two independent intensive property will be enough to determine the other properties.

يعني اي خاصيتين intensive و independent
سيحددان باقي خاصائص الماده في الحاله المراده
والمعناها.

Intensive properties



$$h = Pv + u$$

* h is enthalpy and it has two meanings

* one is for closed system which is **heat content**

* The other is for open system and it will be discussed later.

Important notes:-

* P & T are dependent in the mixture state

* At P we have T_{sat} and at T we have P_{sat}

* So P & T are not enough to determine the other state's properties

* First, we will discuss state postulation of pure substances (water & R-134a)

* First step is to determine the state of the substance.

حالة المادة $\Rightarrow$
مختلطة
مكبنة

(compressed, mixture, superheated, saturated)

(is it compressed liquid?
mixture? or superheated
vapor?)

How does it done?

For water:-

we can go to table A-4
or A-5

For R-134a:-

we can go to table A-11
or A-12

* In these tables, we have many rows and columns contain these intensive properties

* And from these tables, we can know the state.

V_f = specific volume of saturated liquid

V_g = specific volume of saturated vapor

$$V_{fg} = V_g - V_f$$

h_f = enthalpy of saturated liquid

h_g = enthalpy of saturated vapor

h_{fg} = latent heat of vaporization.
($h_g - h_f$)

u_f = internal energy of saturated liquid

u_g = internal energy of sat. vapor

$$u_{fg} = u_g - u_f$$

كيفية التحقق وعرفه الحالة :-
 إذا أعطاك مثلا
 $P + T$

لدينا طريقتين :-

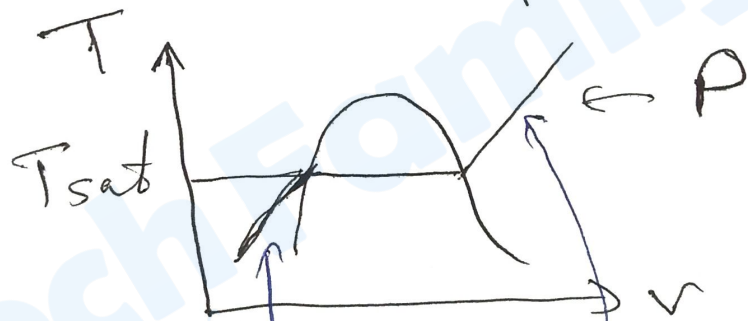
أما تثبيت T أو تثبيت P

لو ثبتنا مثلا P علينا رسم $T-v$ diagram
 والذهاب لجدول P

A-5 for water

or A-12 for R-134a

ثم نجد ال P ونوجد T_{sat} من الجدول



إذا كانت T في السؤال أقل من T_{sat}
 ف نحن في حالة Compressed liquid ($T < T_{sat}$)

إذا كانت T أعلى من T_{sat}
 ف نحن في حالة super heated ($T > T_{sat}$)

$$T = T_{sat}$$

نحن إما
 sat. liquid or mixture or
 sat. vapor

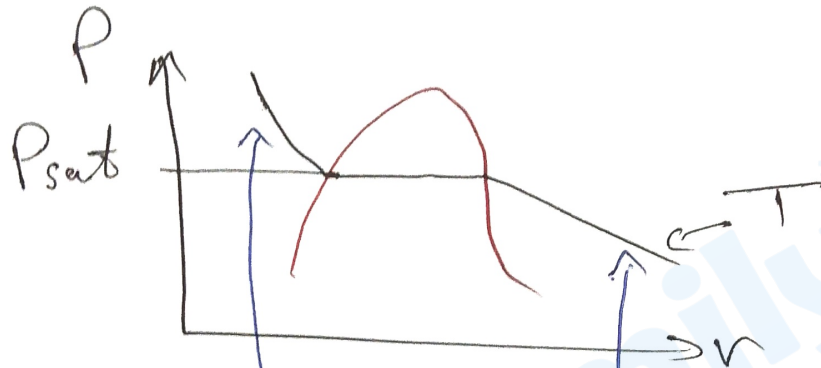
وإذا تبنا T فعلنا $P-v$ diagram

وعلينا استخدام جدول T

A-4 for water

A-12 for R-134a

ثم نجد T في الجدول ونوجد P_{sat}



إذا كان

$P > P_{sat}$

فمنطقة compressed

وإذا كان

$P < P_{sat}$

فمنطقة superheated

وإذا كان $P = P_{sat}$

فمنطقة sat. liquid or mixture or sat. vapor

sat. liquid or mixture or sat. vapor

* But if P or T is given
with (v or u or s or h)

* If P then we have to use
pressure table

A-5 for water

A-12 for R-134

* If T then we have to
use temperature table

A-4 for water

A-11 for R-134a

After determining the table
and the T or P

that is given in
the question:-

we look at the other
property (u, v, h, u)

u for example:-

* if $u < u_f @ T \text{ or } P$

then we are at compressed

* if $u_f < u < u_g$

we are at mixture

* if $u > u_g$

we are at superheated

Some examples:- (saturated liquid or vapor)

- 1- A rigid tank contains 50 kg of saturated liquid water at 90°C . Find pressure and volume of the tank

It's sat. liquid

$$\text{so } V = v_f$$

$$\text{and } P = P_{\text{sat}} @ 90^\circ\text{C}$$

It's water

so we can go to table

A-4

$$P_{\text{sat}} @ 90 = 70.183 \text{ kPa}$$

$$\text{+ } v = v_f = 0.001036 \text{ m}^3/\text{kg}$$

$$\text{so } V = v \cdot m$$

$$= 0.001036 \times 50$$

$$= 0.0518 \text{ m}^3$$

* We have to use table A-5 because P is given.

- 2- A mass of 200 g of sat. liquid water is completely vaporized

at a constant pressure of 100 kPa

find change of volume and the amount of energy transferred to the water

It was sat. liquid and it's completely vaporized!

so in state 2, it's

sat. vapor

state 1:-

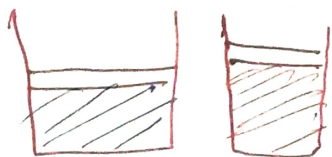
$$V_1 = v_f, m = 0.001043 \times 0.2$$

$$= 0.0002086 \text{ m}^3$$

state 2:-

$$V_2 = v_g, m = 1.6941 \times 0.2$$

$$= 0.33882 \text{ m}^3$$



at 100 kPa

$$\Delta V = V_2 - V_1 = 0.33882 - 0.0002086 \\ \approx 0.3386 \text{ m}^3$$

Amount of energy $\equiv$ latent heat

$$\equiv h_{fg}$$

$$\equiv h_{fg} m = 0.2 \times 2257.5$$

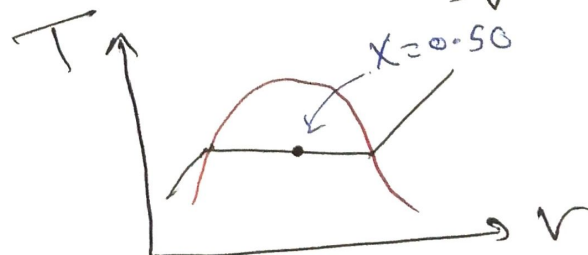
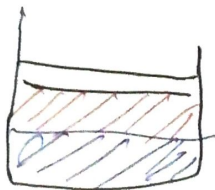
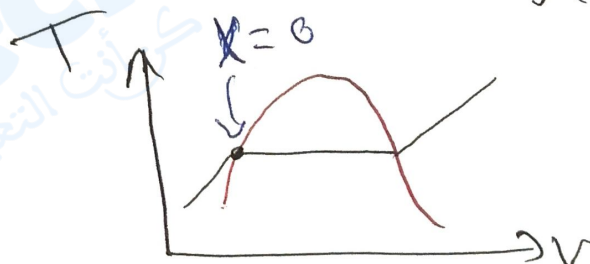
$$\approx 451.5 \text{ kJ}$$

at
100 kPa

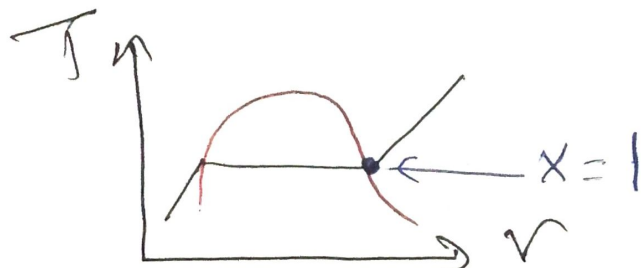
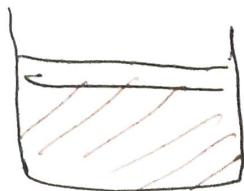
Quality (x):-

It's the fraction of vapor contained in the system (mixture)

$$x = \frac{m_v}{m_v + m_f} \quad \begin{array}{l} \leftarrow \text{mass of vapor} \\ \leftarrow \text{mass of liquid} \end{array}$$



sat.
vapor $\rightarrow$



For mixtures:-

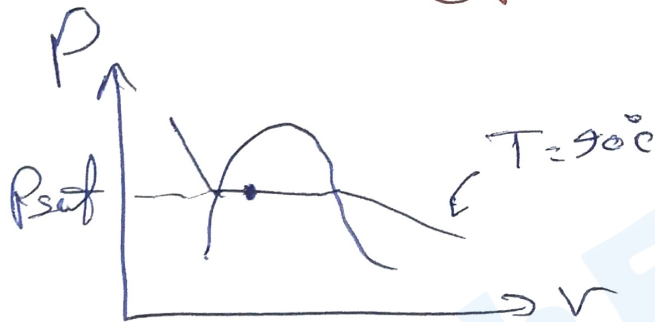
$$h = h_f + x h_{fg}$$

$$u = u_f + x u_{fg}$$

$$v = v_f + x v_{fg}$$

Some examples:-

- 1 - A rigid tank contains 10 kg of water at 90°C, if 8 kg is in the liquid form and the rest is vapor find:-
① pressure ② Volume of the tank



$P = P_{sat} @ 90^\circ\text{C}$
because it's mixture

so $P = 70.183 \text{ kPa}$
(from table A-4)

$$m_f = 8 \text{ kg}$$

$$m_v = 10 - 8 = 2 \text{ kg}$$

$$x = \frac{m_v}{m_f + m_v} = \frac{2}{8 + 2} = 0.2$$

$$v = v_f + 0.2(v_{fg})$$

$$v = 0.001036 + 0.2(2.3893) = 0.47322 \text{ m}^3/\text{kg}$$

$$V = m v = 4.7322 \text{ m}^3$$

2- An 80L vessel contains 4kg of R-134a at a pressure of 160kPa find:-

① Temperature

② quality

③ Enthalpy

④ volume occupied by the vapor.

First, we have to fix the state, -
given

$$P = 160 \text{ kPa}, V = 80 \times 10^{-3} = 0.08 \text{ m}^3$$

for R-134a

$$v = \frac{V}{m} = \frac{0.08 \text{ m}^3}{4 \text{ kg}} = 0.02 \text{ m}^3/\text{kg}$$

$\therefore P$ is given, then we have to go to table A-12

At $P = 160 \text{ kPa}$ ($T = -15.6^\circ\text{C}$)

$$v_f = 0.0007435 \text{ m}^3/\text{kg}$$

$$v_g = 0.12355 \text{ m}^3/\text{kg}$$

$$v = 0.02 \text{ m}^3/\text{kg}$$

$$v_f < v < v_g$$

so we are in mixture

so we have to apply:-

$$0.02 = 0.0007435 + x(0.12355 - 0.0007435) \Rightarrow v = v_f + x v_{fg}$$



$$x = 0.157$$

$$h = h_f + x h_{fg}$$

$h_f + h_{fg}$ is found from table

A-12 at $P = 160 \text{ kPa}$

$$h = 64.1 \text{ kJ/kg}$$

For volume of the vapor:-

$$V_g = m_g v_g$$

$$m_g = X m_{\text{total}} = 4 \times 0.157$$

$$= 0.6284 \text{ g}$$

$$V_g = 0.628 \times 0.12355 \pm 0.0776 \text{ m}^3$$

For compressed or subcooled liquid:-

If the state was compressed:-

For water

if $P \geq 5 \text{ MPa}$
we have to
use table
A-7

if $P < 5 \text{ MPa}$
we have
to use
table A-4
A-4 & not
A-5 !!!
just A-4
, okay?

so that
 $h = h_f$
 $v = v_f$
 $u = u_f$
 $s = s_f$

For
R-134a

we have
always

to
use

A-11

(not
A-12)

so that

$$h = h_f$$

$$v = v_f$$

$$s = s_f$$

$$u = u_f$$

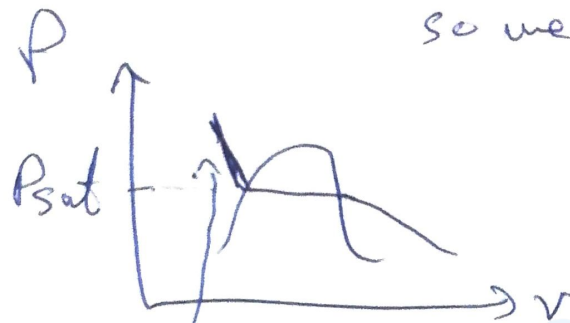
Example :-

Determine the internal energy of water at 5 MPa and 80°C

First, we have to fix the state

we can fix T

so we have to use table A-4



$$P_{\text{sat}} = 47.416 \text{ kPa}$$

$$P > P_{\text{sat}}$$

So it's compressed liquid

$$\therefore P = 5 \text{ MPa}$$

so we can use table A-7

At 5 MPa and 80°C

$$u = 333.82 \text{ kJ/kg.}$$

If the state is superheated

we have to transfer to:-

Table A-6 for water

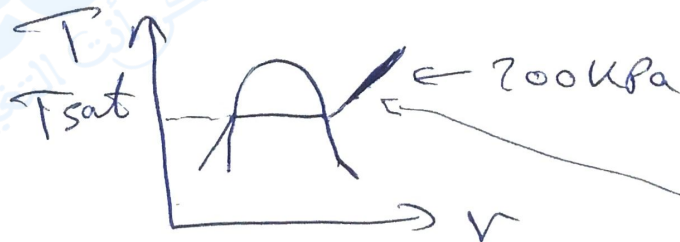
Table A-13 for R-134a

Example:-

Determine the internal energy of water at 200 kPa and 300°C

First, we have to fix the state

we can fix $P = 200 \text{ kPa}$



from pressure table:-

(A-5)

$T_{\text{sat}} = 120.21^\circ\text{C}$
at 200 kPa

$T > T_{\text{sat}}$

so the water is superheated vapor
so we have to move to table
A-6

$u @ 200 \text{ kPa}, 300^\circ\text{C} = 2808.8 \text{ kJ/kg}$

Ideal gas :-

الغاز المثالي هو غاز يميل إلى أن يكون ضغطه منخفض ودرجة حرارته عالية وتكون جزيئاته منتشرة في الفراغ وبمساحة كبيرة
وبالتالي كلما زادت درجة حرارة الغاز وزيادته

reduced pressure $\rightarrow P_r = \frac{P}{P_{cr}}$

reduced temperature $\rightarrow T_r = \frac{T}{T_{cr}}$

P_{cr} & T_{cr} are in table A-1 for different gases

شروط الغاز المثالي :-

1- $P_r < 1$

2- $T_r > 2$

* In thermodynamics 1, we assume that the gas is ideal in all problems and questions.

Ideal gas equations:-

$$P v = R T$$

m^3/kg

$$P V = m R T$$

m^3

$$P V = N R_u T$$

moles

mass

$$N = \frac{m}{M}$$

molar mass

it's found in table A-1 for different gases

kg/mol

$$R_u = 8.31447 \text{ kJ/kmol}\cdot\text{K}$$

R is the gas constant and it's found in table A-1 for different gases

End of chapter 3

CHAPTER 4

ENERGY ANALYSIS OF CLOSED SYSTEMS

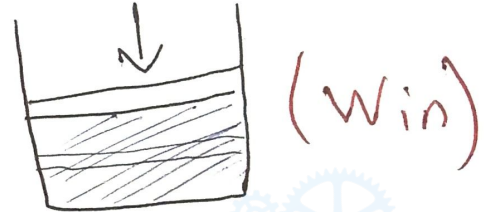
Moving boundary work :-

It's the work associated with the moving of piston in piston cylinder device.

* بحيث ان حركة المكبس
للأسفل (compression) تضيق

للمائع طاقة تخزن فيه

(work done on system by surrounding)

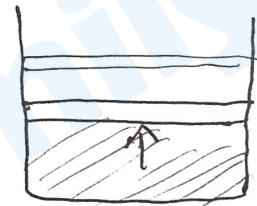


(Win)

* وحركه المكبس للأعلى (expansion)

نتج من استخدام المائع لطاقته
الداخلية لتتمكن جزيئاته من دفع
المكبس للأعلى

(work done by system on surrounding)



(Wout)

Moving boundary work (Wb) :-

$$W_b = \int_{v_1}^{v_2} P dv \quad (\text{KJ/kg})$$

$W_b \equiv$ Area under P-v diagram.

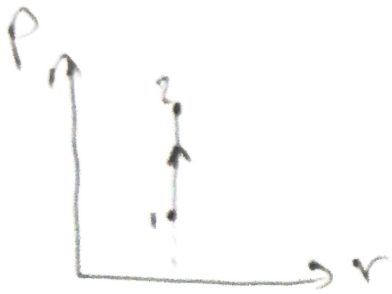
Moving boundary work for different processes:-

1 - isochoric (isometric) process:-

$$V_1 = V_2$$

$$\text{So } W_b = \int_{V_1}^{V_2} P dV = 0$$

So $W_b = 0$ for isochoric process (piston is fixed)



2 - isobaric process:-

$$W_b = \int_{V_1}^{V_2} P dV$$

$P \equiv \text{constant}$

$$\text{So } W_b = P \int_{V_1}^{V_2} dV = P(V_2 - V_1)$$

So $W_b = P \Delta V$ for isobaric process

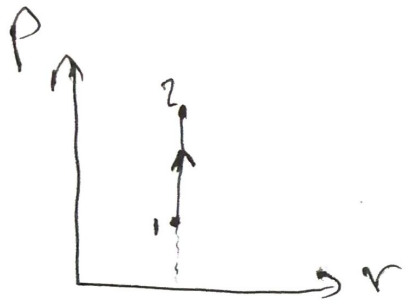
Moving boundary work for different processes:-

1 - isochoric (isometric) process:-

$$V_1 = V_2$$

$$\text{So } W_b = \int_{V_1}^{V_2} P dV = 0$$

So $W_b = 0$ for isochoric process (piston is fixed)



2 - isobaric process:-

$$W_b = \int_{V_1}^{V_2} P dV$$

$P \equiv \text{constant}$

$$\text{So } W_b = P \int_{V_1}^{V_2} dV = P(V_2 - V_1)$$

So $W_b = P \Delta V$ for isobaric process

3 - Isothermal process of an ideal gas:-

for ideal gas when T is constant:-

$$P_1 V_1 = P_2 V_2$$

so $PV = C$ (constant)

$\leftarrow$ $= P_1 V_1$ OR $P_2 V_2$

$$W_b = \int_{V_1}^{V_2} P dV$$

$$P = \frac{C}{V}$$

$$W_b = \int_{V_1}^{V_2} \frac{C}{V} dV = C \int_{V_1}^{V_2} \frac{1}{V} dV$$

$$W_b = C \ln \frac{V_2}{V_1}$$

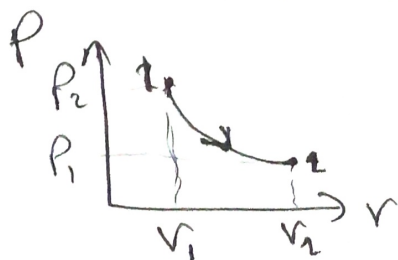
so that $C = P_1 V_1$ OR $C = P_2 V_2$

4 - Polytropic process:-

$$P V^n = \text{constant}$$

$$P_1 V_1^n = P_2 V_2^n$$

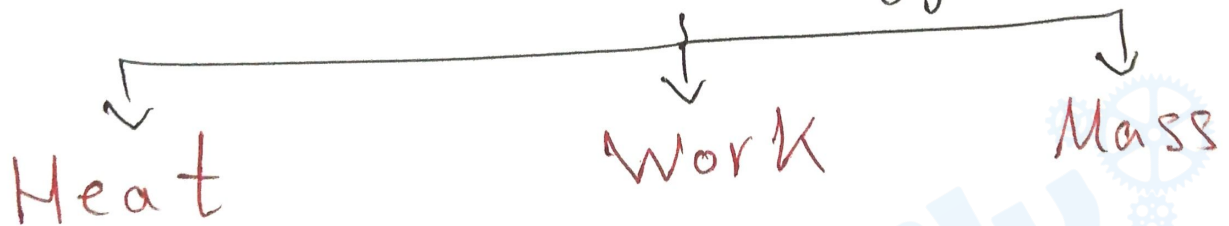
$$W_b = \frac{P_2 V_2 - P_1 V_1}{1 - n} = \frac{mR(T_2 - T_1)}{1 - n}$$



First law of thermodynamics:-

* Energy can be converted from one form to another, but the total amount of energy remains constant.

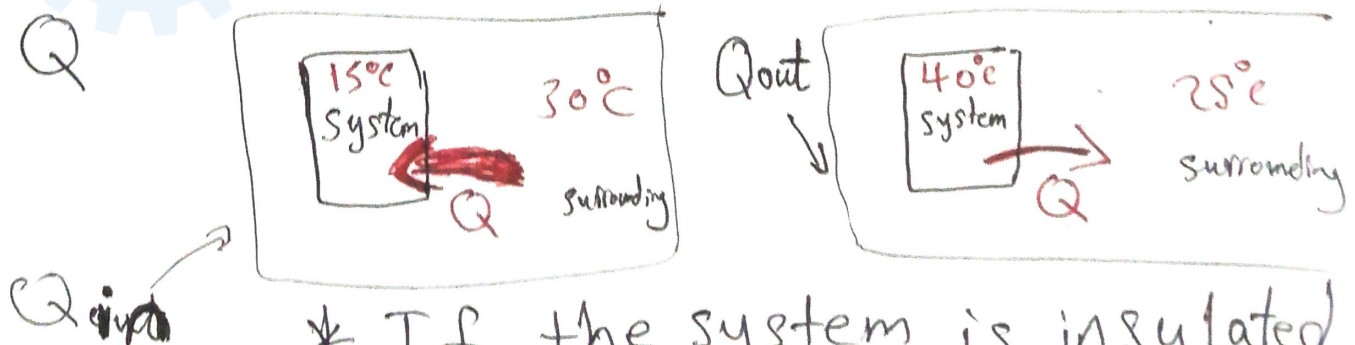
Forms of energy interactions



1 - Heat:-

* Heat is a form of energy and it's represented by Q

* When there is a temperature difference between the system and the surrounding, there will be heat transfer from the hot medium to the cold one



* If the system is insulated, so there will not be heat transfer ($Q = 0$)

2- Work

Work Types

Boundary work

- * Just if the system is piston cylinder
- * if expansion it's Wout
- * if compression it's Win

Others

- Electric work (like heater)
- shaft work
- Baddle work
- * They are all Win if the system is closed

$$W_{\text{electric}} = I V t$$

* Win if the work is done on system

* Wout if the work is done by system.

3- Mass

* When an amount of mass transfers to or from a system, an amount of energy will be transferred ~~to~~ or from the system

* If the system is closed, so no mass can transfer to or from it, so no energy will be added or removed from the system by mass

Energy balance equation (first law) formula

$$\begin{array}{c} \text{energy} \\ \text{Entered} \\ \text{the} \\ \text{system} \end{array} \rightarrow E_{in} - E_{out} = \Delta E_{\text{system}}$$

$\uparrow$ energy
leaved
the
system

$\uparrow$ The change
in the
energy content
of the
system

* If ΔE_{system} is $\oplus \Rightarrow E_{in} > E_{out}$
 $\Rightarrow$ means energy added to the system

* If ΔE_{system} is $\ominus \Rightarrow E_{\text{in}} < E_{\text{out}}$
 $\Rightarrow$ Means that energy is lost.

$$\Delta E_{\text{system}} = \Delta U + \Delta P.E + \Delta K.E$$

Changes in
kinetic and
potential
energies.

so

$$E_{\text{in}} - E_{\text{out}} = \Delta U + \Delta P.E + \Delta K.E$$

* In thermodynamics, we always deal
with stationary systems (ثابتة)
so $\Delta P.E + \Delta K.E = 0$

$$E_{\text{in}} - E_{\text{out}} = \Delta U$$

$$\begin{aligned} & (W_{\text{in}} + Q_{\text{in}} + \sum (\text{Energy by mass})_{\text{in}}) \leftarrow E_{\text{in}} \\ & - (W_{\text{out}} + Q_{\text{out}} + \sum (\text{Energy by mass})_{\text{out}}) \leftarrow E_{\text{out}} \\ & = \Delta U \end{aligned}$$

If the system is closed :-

$$\sum (\text{Energy by mass})_{\text{in}} - \sum (\text{Energy by mass})_{\text{out}} = 0$$

So the energy balance equation of a closed system becomes :-

$$(Q_{\text{in}} + W_{\text{in}}) - (Q_{\text{out}} + W_{\text{out}}) = \Delta U$$

Specific Heats :-

specific heat :- It's the amount of heat needed to raise the temperature of 1kg of the substance by 1 Kelvin.

الحرارة النوعية :- كمية الحرارة اللازمة لرفع درجة حرارة
كجم واحد من مادة معينة درجة واحدة.

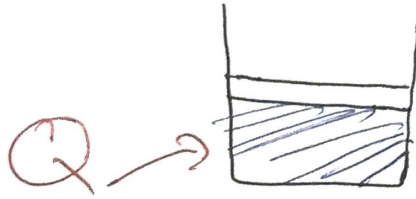
* كل مادة لها نوعان من الحرارة النوعية

* We have two types of specific heat :-



C_p :- It represents the amount of heat that we have to add at constant pressure to increase the T by 1 K

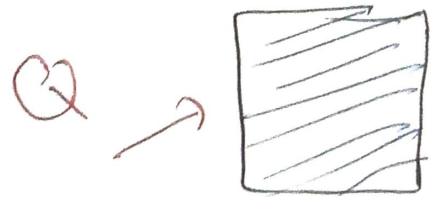
كمية الحرارة المضافة
إضافتها ~~إلى~~
بنفس ثابت لرفع
درجة حرارة المادة
درجة واحدة



$P \equiv \text{constant}$
because
 P_{atm} and
the weight
of the
piston are
constant

C_v :- It represents the amount of heat that we have to add at constant volume to increase the T by 1 K

كمية الحرارة
المضافة
على حجم ثابت
لرفع درجة الحرارة
درجة واحدة



$V \equiv \text{constant}$

$$[C_p \text{ and } C_v] = \text{KJ/kg.K}$$

* C_p and C_v are function of T and they can be found in table A-2 b at different temperatures for different gases and in table A-2 a just at 300 K .

Enthalpy and Internal energy of ideal gases

* First, we have to know that enthalpy and internal energy are function of temperature only for ideal gases

$$C_p = R + C_v$$

$$K = \frac{C_p}{C_v}$$

$$C_p > C_v$$

$$h = h(T) \quad u = u(T)$$

for ideal gases.

$$h = C_p T \quad u = C_v T$$

for ideal gases

$$\Delta h = C_p \Delta T \quad \Delta u = C_v \Delta T$$

which C_v & C_p that we have to use in these equations??

It depends on the question
يعتمد على السؤال

ممكن يطلب فرض ان C_p و C_v ثابتان عن درجه

المحراره في الحالة ① (T_1)

فمن وجدها من الجواب اول

وفي بعض الاسئلة يطلبها عند 300K

مختوجها من جدول A-2a

وفي بعض الاسئلة (الغالب) علينا إيجاد $C_{Paverage}$

يعني إما متوسط

$$T_{average} = \frac{T_1 + T_2}{2}$$

ثم نوجد C_P و C_V عندها من

جدول A-2b

او نوجد C_P و C_V عند T_1 وعند T_2

$$C_{Vaverage} = \frac{C_{V1} + C_{V2}}{2}$$

$$C_{Paverage} = \frac{C_{P1} + C_{P2}}{2}$$

الطريقة الاخرى لإيجاد h او U عند

أي حرارة هي من جداول اللغزات

A-22 To A-17

فهذه الجداول تعطي h و U عند

كل حرارة

لكن تنبيه ان من A-18 الى A-22

تكون h و U بوحدة kJ/kmol

molar
mass

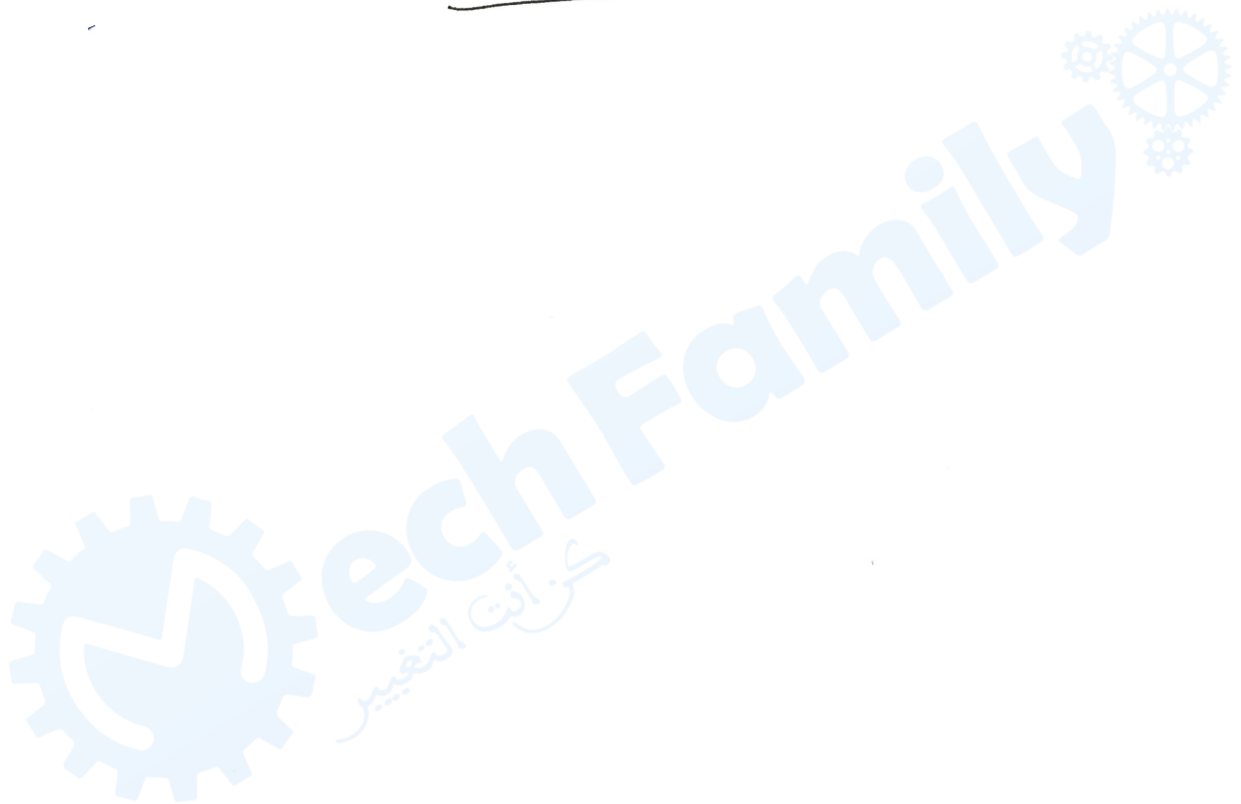


At A-1

قواعد ان تقع على ال

لتحويلها الى kJ/kg

End of chapter 4



CHAPTER 5

MASS AND ENERGY ANALYSIS OF CONTROL VOLUMES

Mass and volume flow rate:-

Mass flow rate:- The amount of mass flowing through a cross section per unit time

عدد الكيلوجرامات التي تقطع (تتدفق) خلال مقطع معين خلال زمن معين

لو فرضنا ان لدينا ماسوره لها مساحة مقطعية معينة A_c ، وكان هناك مائع له كثافه ρ تتدفق خلال هذه الماسوره بسرعه متوسطه V_{avg}

يكون المعدل الزمني لتدفق هذا المائع (mass flow rate) :-

$$\text{mass flow rate} \rightarrow \dot{m} = \rho V_{avg} A_c$$

kg/m^3 m/s m^2

$$[\dot{m}] = \text{kg/s}$$

كم kg يمر كل ثانيه

If we have a circular cross section:-

$$A_c = \frac{\pi}{4} D^2$$



Volume flow rate:-

It's the volume of the fluid flowing through a cross section per unit time

كم متر مكعب - متحركاً المقطع خلال زمن معين

$$m^3/s \rightarrow \dot{V} = V_{avg} A_c$$

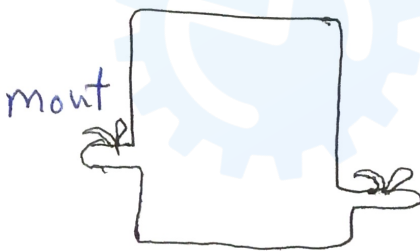
$\uparrow$ m/s $\uparrow$ m^2

So

$$\dot{m} = \rho \dot{V}_{avg} A_c$$
$$\dot{m} = \rho \dot{V}$$

Conservation of mass principle:-

$$\dot{m}_{in} - \dot{m}_{out} = \Delta \dot{m}_{system}$$



$$\dot{m}_{in} - \dot{m}_{out} = \Delta \dot{m}_{system}$$

$\uparrow$ معدل المدخل $\uparrow$ معدل المخرج

معدل تغير كتلة النظام
(مثلاً كتلة المياه بجزء بحدود - أو بطل)

Mass balance of steady flow devices:-

Steady flow devices:-

هناك نوعان من control volumes والتي تتميز بالعدد من الخصائص منها :-

$$\sum \dot{m}_{in} = \sum \dot{m}_{out}$$

$$\dot{m}_{in} = \dot{m}_{out} \quad \Delta \dot{m}_{system} = 0$$

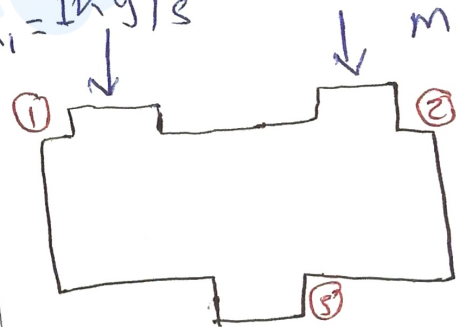
* في هذه الأجهزة هناك كتل تدخل وتخرج بنسب الوقت، وجميع الكتل الداخلة = مجموع الكتل الخارجة
 $\dot{m}_1 = 1 \text{ kg/s}$ $\dot{m}_2 = 3 \text{ kg/s}$

$$\sum \dot{m}_{in} = \sum \dot{m}_{out}$$

$$\dot{m}_{in} = \dot{m}_{out}$$

$$\rho_1 v_1 A_1 = \rho_2 v_2 A_2$$

Single ↑ Stream (مدخل واحد وخرج واحد)



$$\dot{m}_3 = \dot{m}_1 + \dot{m}_2 = 1 + 3 = 4 \text{ kg/s}$$

$$\left(\rho_1 v_1 A_1 + \rho_2 v_2 A_2 + \rho_3 v_3 A_3 + \dots \right)_{in} = \left(\rho_1 v_1 A_1 + \rho_2 v_2 A_2 + \dots \right)_{out}$$

Special case (Incompressible flow)

* The incompressible fluids like liquids, have constant densities.

$$\rho \approx \text{constant}$$

so if an incompressible fluid flows through a steady flow device:-

$$\dot{m}_1 = \dot{m}_2$$

$$\rho_1 v_1 A_1 = \rho_2 v_2 A_2$$

$$\rho_1 \approx \rho_2$$

$$v_1 A_1 = v_2 A_2$$

$$\dot{V}_1 = \dot{V}_2$$

$$\sum \dot{V}_{in} = \sum \dot{V}_{out}$$

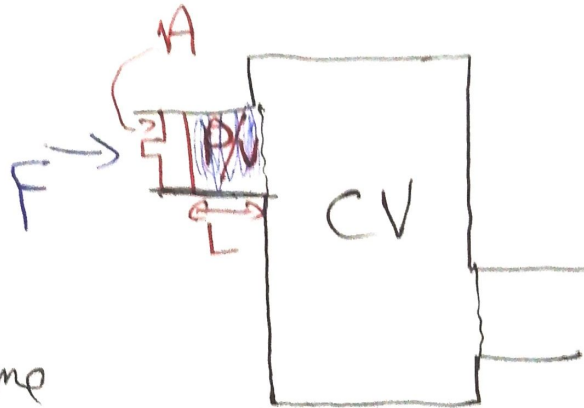
Flow work :-

$$W = FL$$

$$W = PAL$$

$$W = PV$$

↑ flow work ↑ Pressure ↑ Volume



Any flowing fluid carries a type of energy called flow energy

Total energy carried by any flowing fluid :-

$$E_{\text{mass (fluid)}} = W_{\text{flow}} + U + K.E + P.E$$

$$E_{\text{mass (fluid)}} = PV + U + \frac{1}{2}mV^2 + mgh$$
$$Pv + U = h$$

$$E_{\text{mass (fluid)}} = h + K.E + P.E$$

$$\dot{E}_{\text{mass}} = \dot{m} (h + P.E + K.E)$$

rate of energy transferred to or from a system by mass (Power)

kJ/kg kJ/kg kJ/kg

$P.E = gh$
 $K.E = \frac{1}{2} v^2$

$(\text{kJ/s} = \text{kW})$

Open systems
(control volumes)

Steady flow systems unsteady flow systems

1 - steady flow systems:-

$$\dot{m}_1 = \dot{m}_2$$

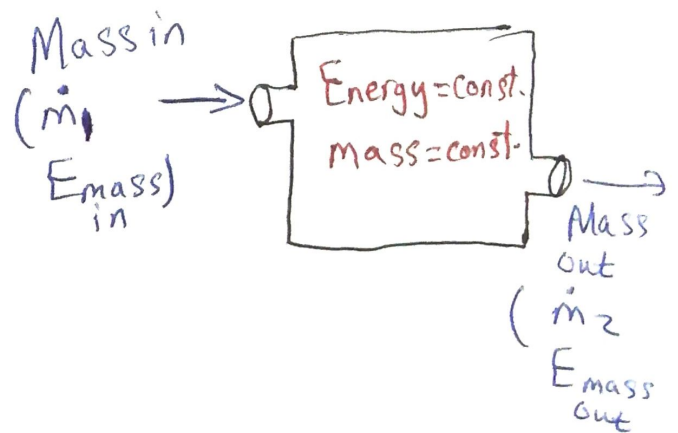
$$\rho_1 v_1 A_1 = \rho_2 v_2 A_2$$

$$\Delta \dot{E}_{\text{system}} = 0$$

$$\dot{E}_{\text{in}} - \dot{E}_{\text{out}} = 0$$

$$\dot{E}_{\text{in}} = \dot{E}_{\text{out}}$$

$$(\dot{Q}_{\text{in}} + \dot{W}_{\text{in}} + \sum \dot{E}_{\text{mass in}}) = (\dot{Q}_{\text{out}} + \dot{W}_{\text{out}} + \sum \dot{E}_{\text{mass out}})$$



$$\sum \dot{E}_{\text{mass in}} = \dot{m}_1 (h_{\text{in}} + K \cdot E_{\text{in}} + P \cdot E_{\text{in}})$$

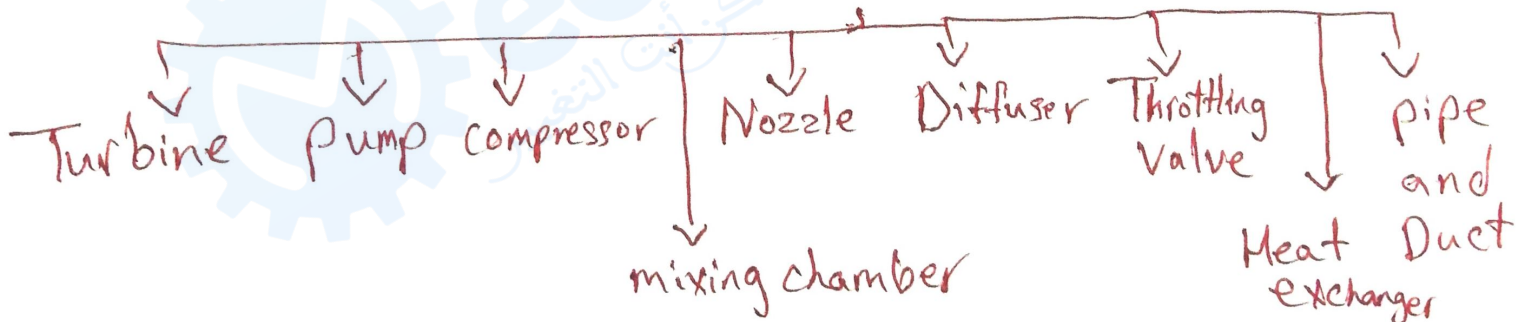
$$\dot{Q}_{\text{in}} + \dot{W}_{\text{in}} + \sum \dot{m} (h_{\text{in}} + K \cdot E_{\text{in}} + P \cdot E_{\text{in}})$$

$$= \dot{Q}_{\text{out}} + \dot{W}_{\text{out}} + \sum \dot{m} (h_{\text{out}} + K \cdot E_{\text{out}} + P \cdot E_{\text{out}})$$

$$(\dot{Q}_{\text{in}} + \dot{W}_{\text{in}}) - (\dot{Q}_{\text{out}} + \dot{W}_{\text{out}}) = \dot{m} (\Delta h + \Delta P \cdot E + \Delta K \cdot E)$$

for $\nearrow$
steady
flow systems
(single inlet and outlet)

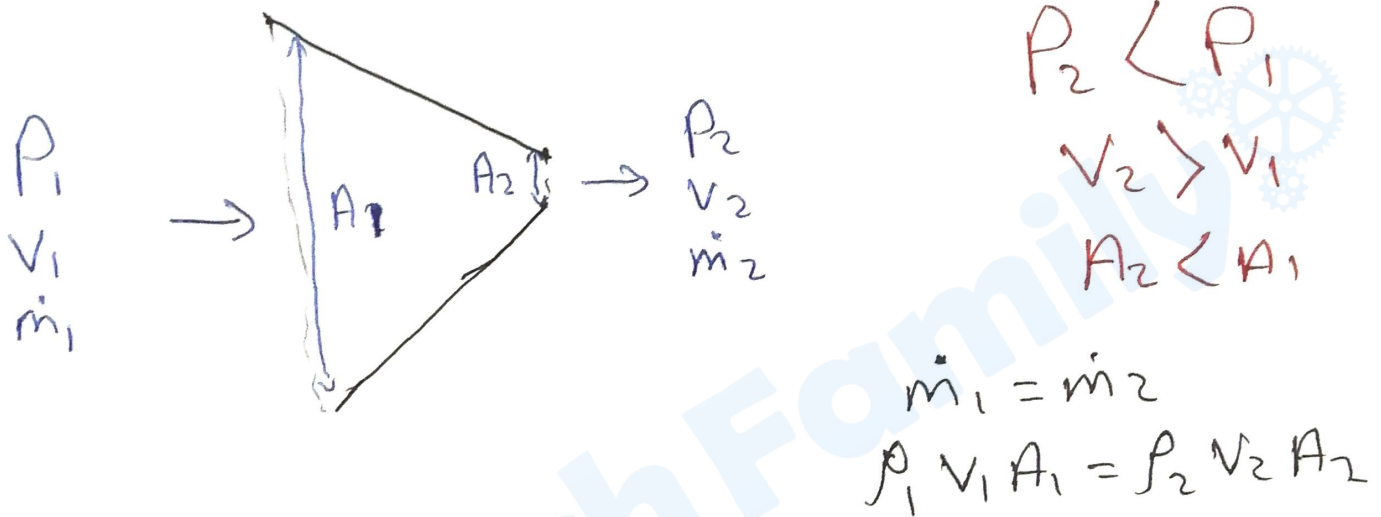
Types of engineering steady flow devices



1- Nozzle and Diffuser :-

Nozzle:-

It's a steady flow device used to increase the velocity of the fluid by decreasing its pressure.



First law:-

$$(\dot{Q}_{in} + \dot{W}_{in}) - (\dot{Q}_{out} + \dot{W}_{out}) = \dot{m} (\Delta h + \Delta P.E + \Delta K.E)$$

mainly

$$\left[\begin{array}{l} \dot{Q}_{in} = \dot{Q}_{out} = 0 \\ \dot{W}_{in} = \dot{W}_{out} = 0 \\ \Delta P.E = 0 \end{array} \right.$$

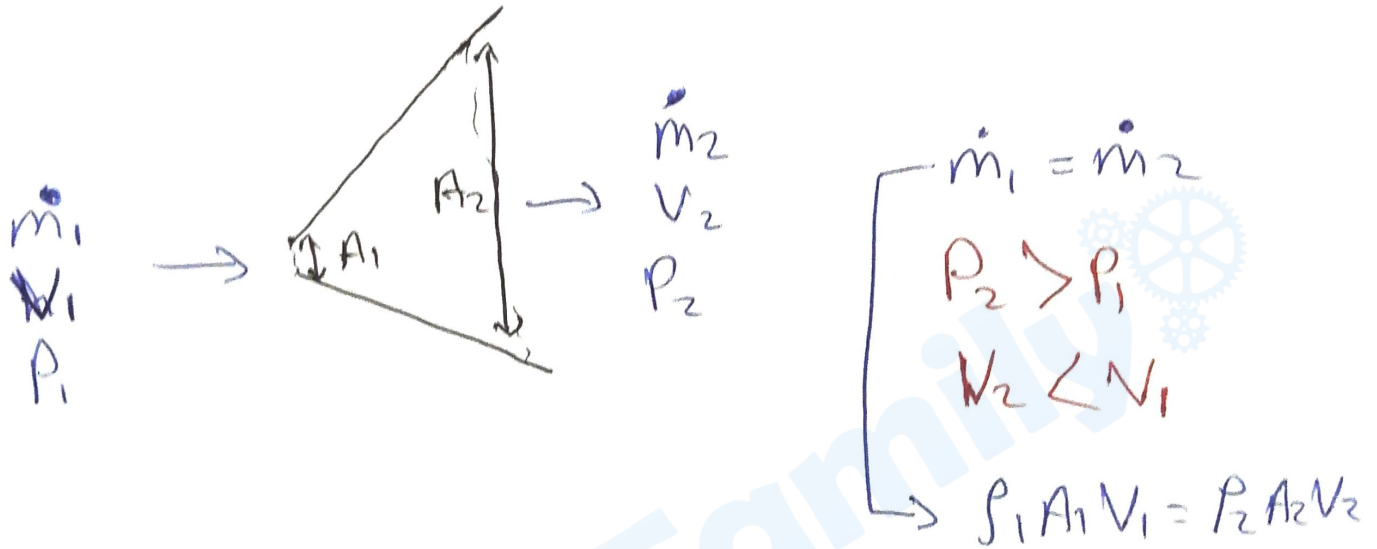
$$0 = \dot{m} (\Delta h + \Delta K.E)$$

$$\dot{m} (-h_2 + h_1) = \dot{m} (K.E_2 - K.E_1)$$
$$-\dot{m} h_2 + \dot{m} h_1 = \dot{m} K.E_2 - \dot{m} K.E_1$$

$$\dot{m} (h_{in} + \frac{1}{2} V_{in}^2) = \dot{m} (h_{out} + \frac{1}{2} V_{out}^2)$$

Diffuser:-

It's a steady flow device that increases the pressure of the fluid by decreasing its velocity



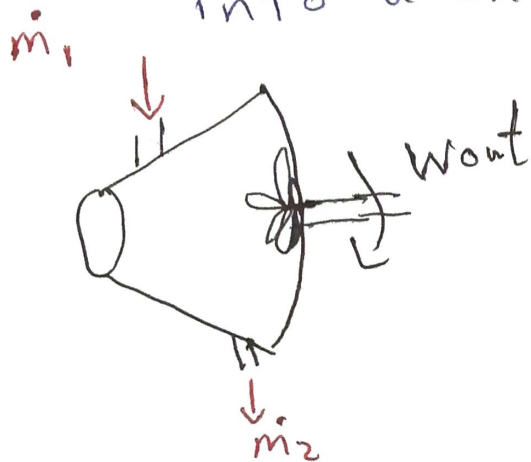
First law:-

$$E_{in} = E_{out}$$

$$\dot{Q}_{in} + \dot{W}_{in} + \dot{m} (h_{in} + K \cdot E_{in} + P \cdot E_{in}) = \dot{Q}_{out} + \dot{W}_{out} + \dot{m} (h_{out} + K \cdot E_{out} + P \cdot E_{out})$$
$$\dot{m} h_{in} + \dot{m} K \cdot E_{in} = \dot{m} h_{out} + \dot{m} K \cdot E_{out} + \dot{m} (P \cdot E_{out} - P \cdot E_{in})$$
$$\dot{m} h_{in} + \frac{1}{2} \dot{m} V_{in}^2 = \dot{m} h_{out} + \frac{1}{2} \dot{m} V_{out}^2$$

2-Turbine:-

It's a steady flow device that converts the energy of the fluid into a shaft work



$$\dot{m}_1 = \dot{m}_2 = \dot{m}$$

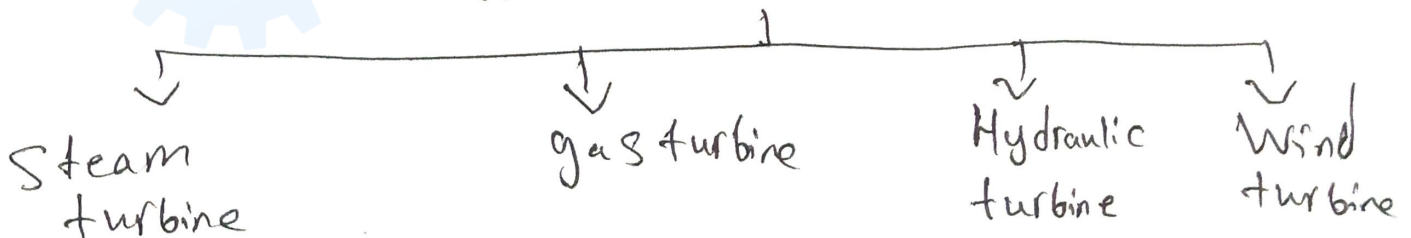
First law analysis:-

$$\dot{E}_{in} = \dot{E}_{out}$$

$$\cancel{\dot{Q}_{in}} + \cancel{\dot{W}_{in}} + \dot{m} (\cancel{k \cdot E_{in}} + h_{in} + \cancel{P \cdot E_{in}}) = \cancel{\dot{W}_{out}} + \cancel{\dot{Q}_{out}} + \dot{m} (\cancel{k \cdot E_{out}} + h_{out} + \cancel{P \cdot E_{out}})$$

$$\dot{m} (h_{in}) = \dot{W}_{out} + \dot{m} (h_{out})$$

we have many types of turbines



In thermodynamics, we just study steam and gas turbines.

3 - Compressor and pump

Compressor :-

It's a device that is used to compress (increase the pressure) of gases by consuming work

$$\dot{E}_{in} = \dot{E}_{out}$$

$$\cancel{\dot{Q}_{in}} + \cancel{\dot{W}_{in}} +$$

$$\dot{m} (\cancel{P.E_{in}} + \cancel{K.E_{in}} + h_{in}) + \cancel{\dot{W}_{in}}$$

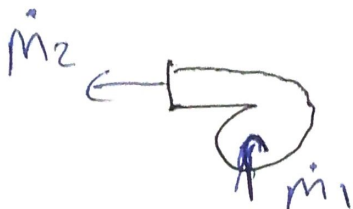


$$= \cancel{\dot{Q}_{out}} + \cancel{\dot{W}_{out}} + \dot{m} (\cancel{P.E_{out}} + \cancel{K.E_{out}} + h_{out})$$

$$\dot{W}_{in} + \dot{m} h_{in} = \dot{m} h_{out}$$

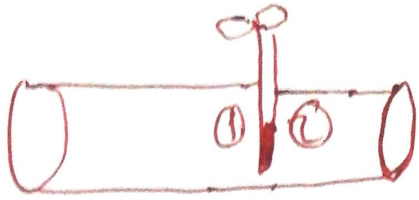
pump :-

It's a device that is used to compress (increase the pressure) of liquids by consuming work



$$\dot{W}_{in} + \dot{m} h_{in} = \dot{m} h_{out}$$

4 - Throttling valve



It's a device that decreases the pressure of the fluid and contains a constant enthalpy process

Energy analysis

$$\dot{E}_{in} = \dot{E}_{out}$$

$$\cancel{\dot{Q}_{in}} + \cancel{\dot{W}_{in}} + \dot{m}(h_{in} + \cancel{P \cdot \dot{E}_{in}} + \cancel{K \cdot \dot{E}_{in}}) = \cancel{\dot{Q}_{out}} + \cancel{\dot{W}_{out}} + \dot{m}(h_{out} + \cancel{P \cdot \dot{E}_{out}} + \cancel{K \cdot \dot{E}_{out}})$$

* It's mainly used in air conditioning applications

$$h_{in} = h_{out}$$

$$P = \text{constant}$$

$$\text{so } T_2 < T_1$$

some types

Adjustable valve



Porous plug



Capillary tube



In ideal gases:-

$$h(T)$$

$$h_1 = h_2$$

so

$$T_1 = T_2$$

5 - Mixing chamber

It's a device that is used to mix fluids.



* An important example

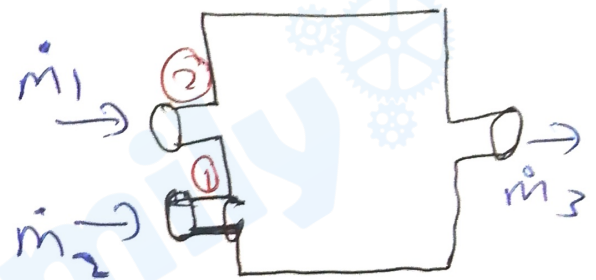
is shower

$$\sum \dot{m}_{in} = \sum \dot{m}_{out}$$

$$\dot{m}_1 + \dot{m}_2 = \dot{m}_3$$

$$\dot{E}_{in} = \dot{E}_{out}$$

$$\begin{aligned} \dot{Q}_{in} + \dot{W}_{in} + \sum \dot{m}_{in} (h_{in} + P \dot{E}_{in} + K \dot{E}_{in}) &= \dot{Q}_{out} + \dot{W}_{out} + \sum \dot{m}_{out} (h_{out} + P \dot{E}_{out} + K \dot{E}_{out}) \end{aligned}$$



$$P_1 \approx P_2 \approx P_3$$

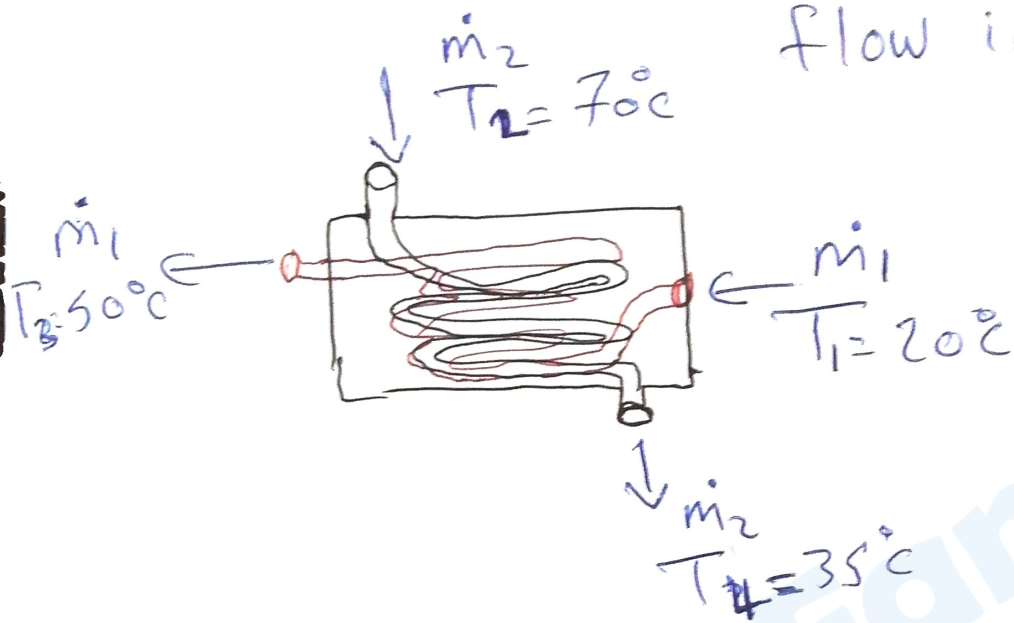
$$\sum \dot{m}_{in} h_{in} = \sum \dot{m}_{out} h_{out}$$

$$\dot{m}_1 h_1 + \dot{m}_2 h_2 = (\dot{m}_1 + \dot{m}_2) h_3$$

$\dot{m}_3$

6-Heat exchanger :-

It's a device that is used to transfer heat between two fluids flow in two streams



$$\dot{m}_1 = \dot{m}_1$$

$$\dot{m}_2 = \dot{m}_2$$

$$\dot{E}_{in} = \dot{E}_{out}$$

$$\cancel{\dot{Q}_{in}} + \cancel{\dot{W}_{in}} + \sum \dot{m} (\cancel{K.E_{in}} + \cancel{h_{in}} + \cancel{P.E_{in}}) = \cancel{\dot{Q}_{out}} + \cancel{\dot{W}_{out}} + \sum \dot{m} (\cancel{K.E_{out}} + \cancel{P.E_{out}} + h_{out})$$

$$\sum_{in} \dot{m} (h_{in}) = \sum_{out} \dot{m} (h_{out})$$

$$\dot{m}_1 h_1 + \dot{m}_2 h_2 = \dot{m}_1 h_3 + \dot{m}_2 h_4$$

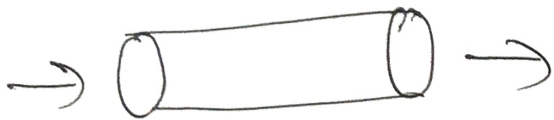
Q between two stream

$$= \dot{m}_1 (h_3 - h_1) = \dot{m}_2 (h_4 - h_2)$$

7- Pipe and duct

They are used to transport fluids.

* The difference between them is that the pipe has a circular cross section, while duct has rectangular cross section



* They can contain any type of energy interactions

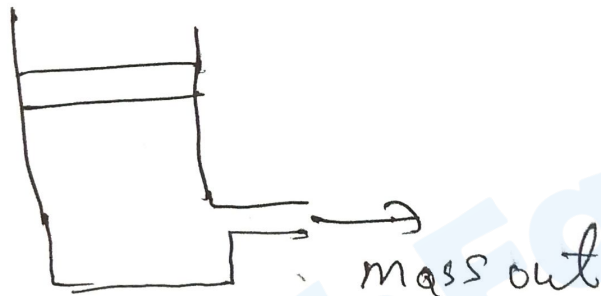
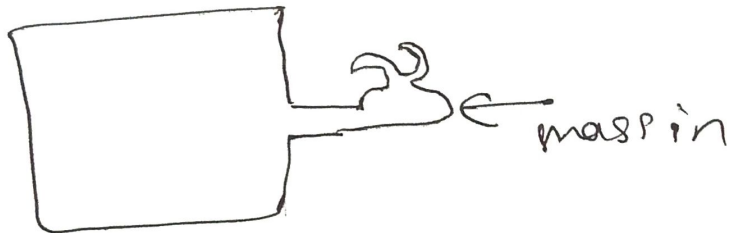
$$\dot{E}_{in} = \dot{E}_{out}$$

$$\dot{Q}_{in} + \dot{W}_{in} + \dot{m} (h_{in} + K \cdot E_{in} + P \cdot E_{in})$$

$$= \dot{Q}_{out} + \dot{W}_{out} + \dot{m} (h_{out} + K \cdot E_{out} + P \cdot E_{out})$$

Unsteady flow system:-

It just contains either
mass in or mass out



$$E_{in} - E_{out} = E_2 - E_1$$

$$m_{in} - m_{out} = \Delta m_{system}$$

$$m_2 - m_1$$

End of chapter 5

CHAPTER 6

THE SECOND LAW OF THERMODYNAMICS

الكفاءة Efficiency

$$\eta = \frac{\text{output} \leftarrow \text{الناتج}}{\text{input} \leftarrow \text{المصروف}}$$

تعبير عن النسبة المستفادة
من الشيء المصروف

$$\Rightarrow \eta = \frac{\text{Heat}}{\text{electrical energy}}$$

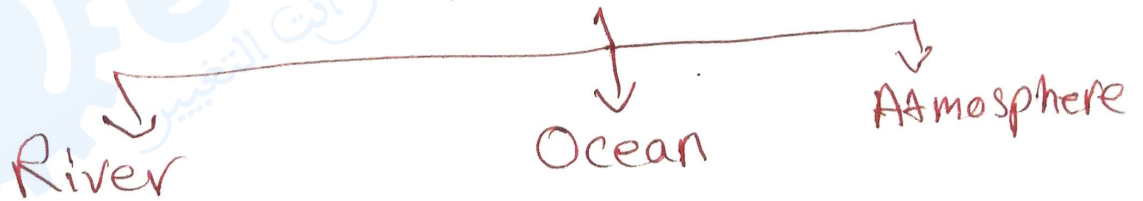
الكفاءة = الناتج / المصروف

Reservoirs:-

They are systems that have very large thermal energy capacity (mass $\times$ specific heat) or heat capacity KJ/K

هي أنظمة لها سعة حرارية كبيرة جدا
بحيث ان مقدار الطاقة الحرارية اللازم لرفع
درجة حرارتها درجة واحدة تكون كبيرة جدا
جدا، وهذه السعة الحرارية الكبيرة ناجية من
كتلتها او حجمها الضخم

Some examples of
reservoirs:-



* مثلاً تسخين ولاء في الجو لن تزيده من
درجة حرارته ابداً

* لن يذوب ثلج ساخن في نهر
لن يؤثر على درجة حرارته

(It's just a latent heat)

Second law of thermodynamics 1-

1 - Any process occur in a certain direction and not in the reverse direction.

Ex: Heat transfers from the hotter medium to the colder one, not the opposite.

2 - Higher quality (like work) energy can be converted totally to lower quality (like heat), but ~~part~~ of lower quality can be converted to higher quality.

3 - No heat engine with $\eta = 100\%$

Ex:-

Electric heater

(input = electric work)

output = heat

so its efficiency $\approx 100\%$

Car engine

(input = heat)

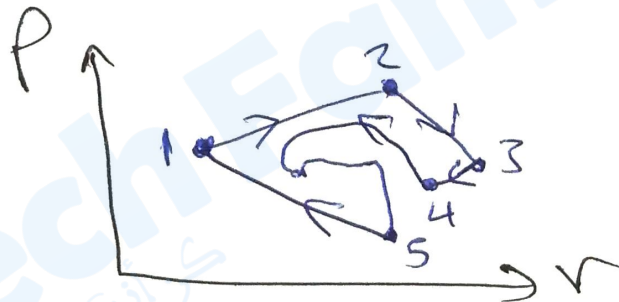
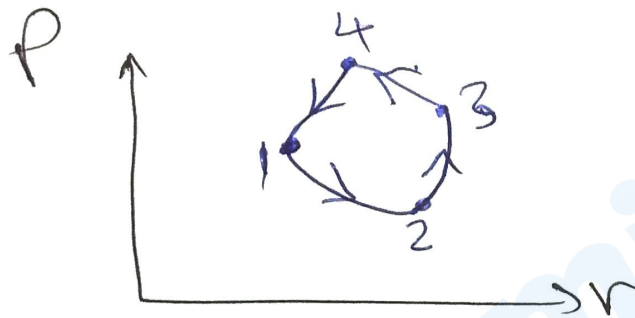
output = electric work

so its efficiency $< 100\%$

Cycle:-

It represents a set of processes done that the first ~~state~~ state = final state

تعبير عن عمليات مغلقة بحيث تكون الحالة الابتدائية = الحالة النهائية



Cycles

Thermodynamic cycles

The working fluid is recycled

in each cycle
(same fluid)

Mechanical cycles

The working fluid is rejected at the end of the cycle and a new fluid enters the cycle when a new cycle starts

Heat engine :-

It's a device that receives heat from a heat source and converts part of it into work and the other part is rejected to a heat sink.

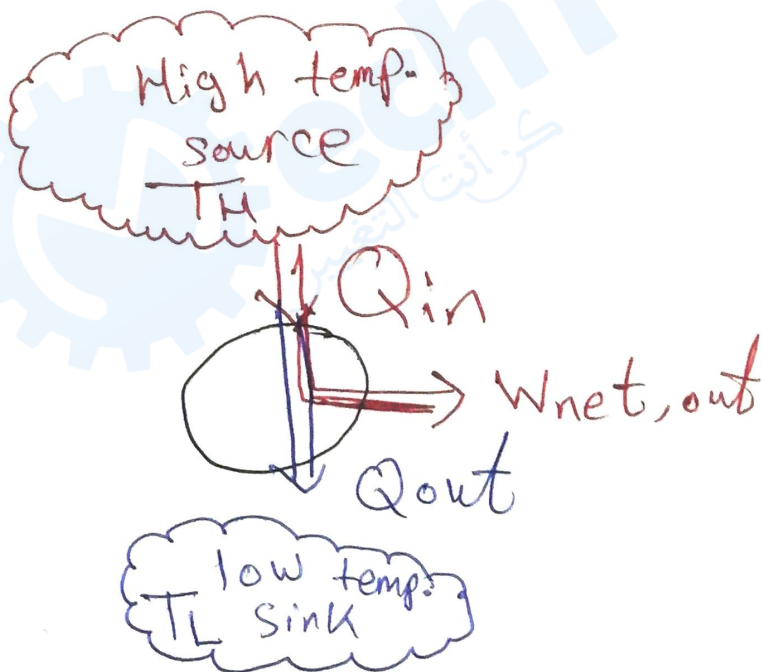
جهاز يستقبل حرارة (طاقة حرارية Q_{in})

من مصدر ذو درجة حرارة عالية ويحول

جزء من هذه الطاقة إلى شغل والجزء الآخر يتم

التخلص منه.

* Heat engine works on thermodynamic cycle



* Car engine is not a heat engine because it works on a mechanical cycle

* Steam power plant is an example of heat engines.

Steam powerplant :-

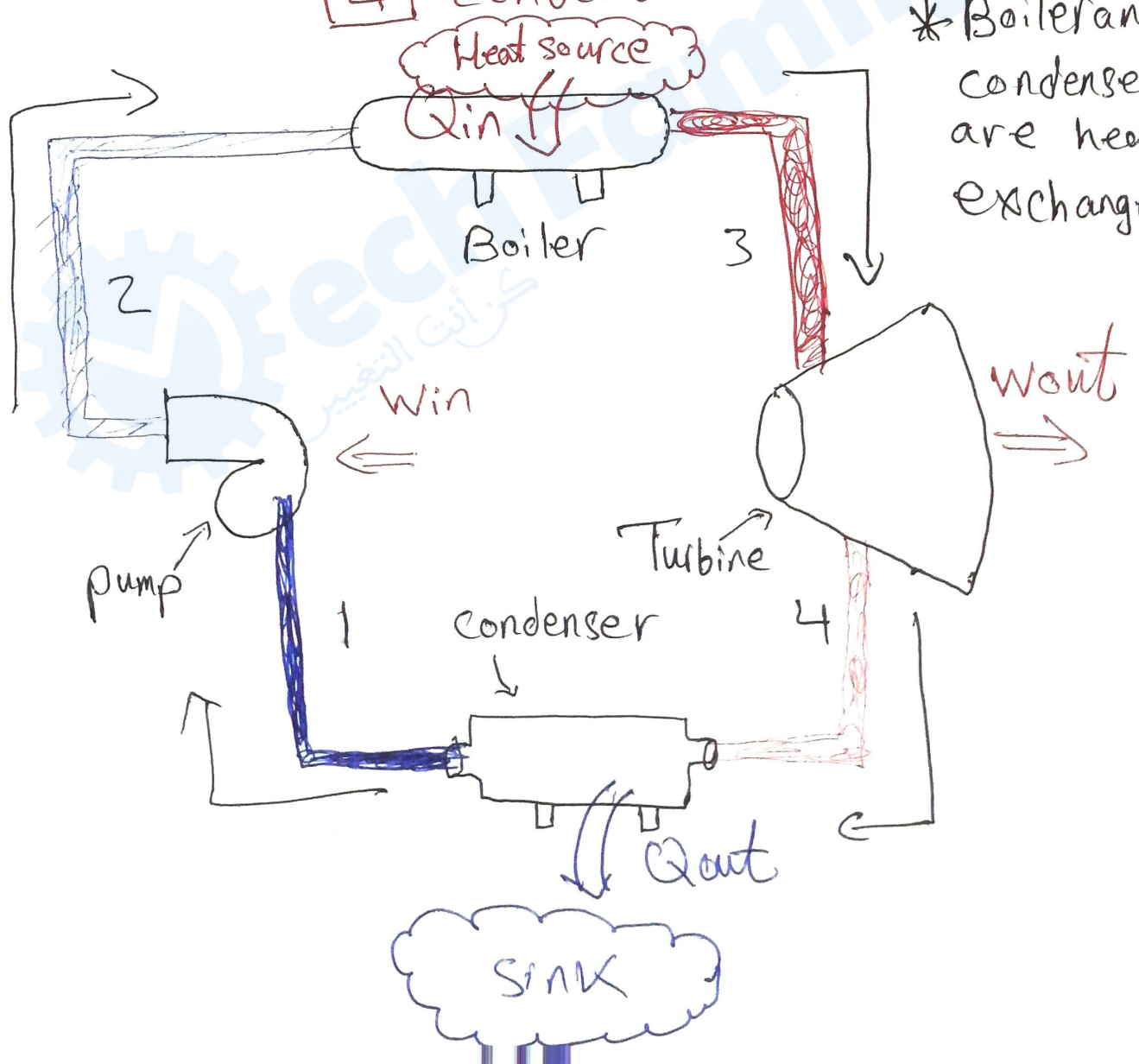
آلة بخارية cycle آلة بخارية
4 عمليات (4 processes)
الآلة البخارية

الآلة البخارية (4 components) :-

- 1 pump
- 2 Boiler
- 3 Turbine
- 4 Condenser

* The working fluid is water.

* Boiler and condenser are heat exchangers



processes :-

(1-2): Compression of a liquid
by a pump (W_{in})

(2-3): Heat addition in
a boiler (Q_{in})

(3-4): The steam do a work
inside the turbine
(W_{out})

(4-1): Heat is rejected by
a condenser.

the cycle is considered
as a closed system:-

$$(Q_{in} + W_{in}) - (Q_{out} + W_{out}) = \Delta U$$

Through a complete
cycle:- $\Delta U = 0$

$$Q_{in} + W_{in} - Q_{out} - W_{out} = 0$$

$$Q_{in} - Q_{out} = W_{out} - W_{in}$$

$$\boxed{Q_{in} - Q_{out} = W_{net}}$$

Thermal efficiency:-

$$\eta_{th} = \frac{\text{output} \leftarrow \text{net}}{\text{input} \leftarrow \text{heat}}$$

$$\eta_{th} = \frac{W_{net}}{Q_{in}}$$

$$\eta_{th} = \frac{Q_{in} - Q_{out}}{Q_{in}}$$

$$\eta_{th} = 1 - \frac{Q_{out}}{Q_{in}}$$

* η_{th} تعبر عن كم الفائدة من الحرارة

التي قدمناها تحول إلى Work

$$Q_{in} = Q_H$$

$$Q_{out} = Q_L$$

High temp. reservoir at T_H

Q_H

H.E $\Rightarrow W_{net, out}$

Q_L

Low temp. reservoir at T_L

$$W_{net, out} = Q_H - Q_L$$

$$\eta_{th} = 1 - \frac{Q_L}{Q_H}$$

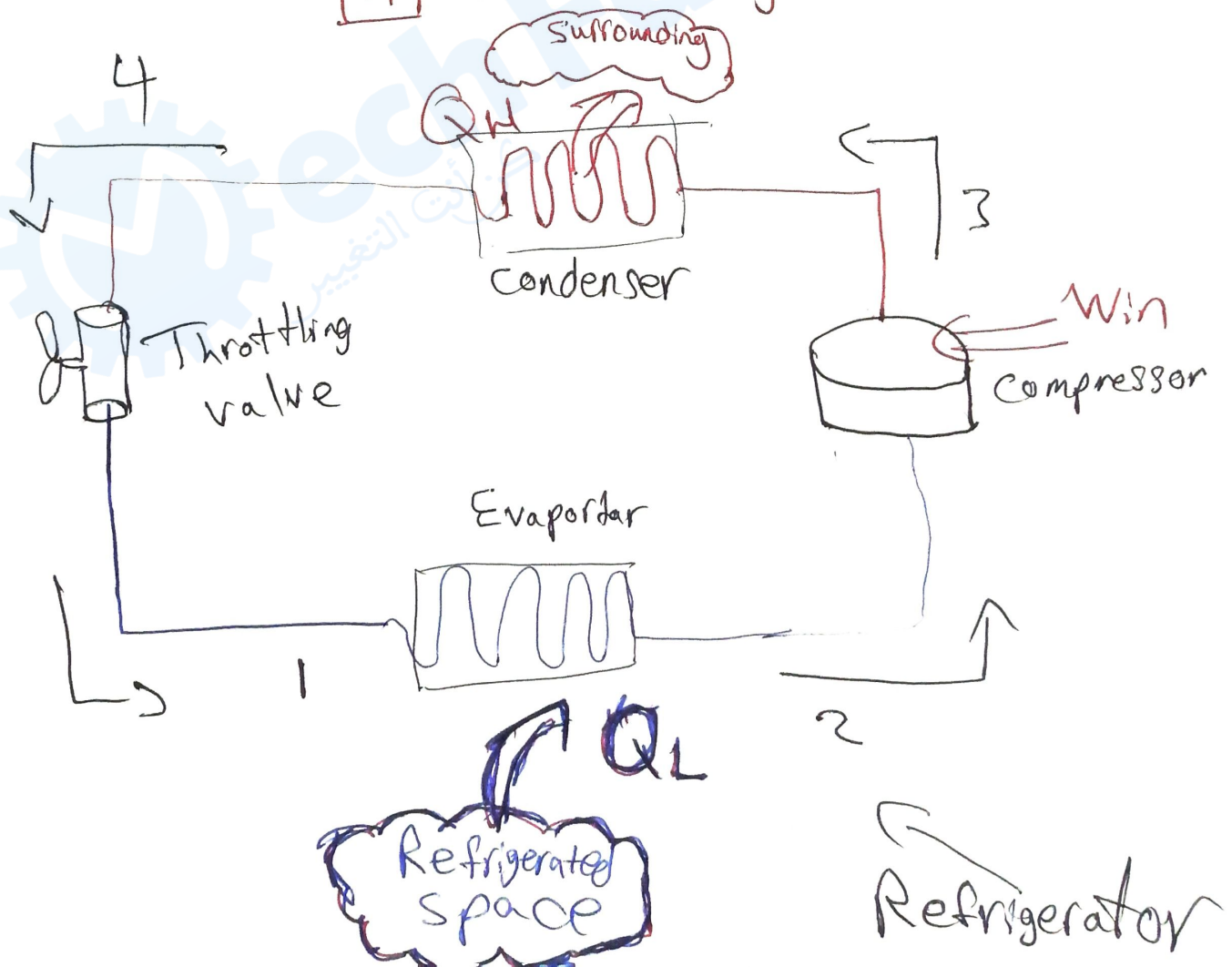
Refrigerators and Heat Pumps:-

* أجهزة التكييف والتبريد والتهوية
بشكل عام تعمل على cycles

* The cycle contains :-

* R-134a
is used
as
a working
fluid

- 1 Evaporator
- 2 compressor
- 3 Condenser
- 4 Throttling valve



Processes:-

(1-2) Absorbing heat from the refrigerated space (Q_L)

(2-3) compressing the fluid by consuming work (W_{in})

(3-4) rejecting the absorbed heat to the surrounding (Q_H)

(4-1) Expansion by a throttle valve

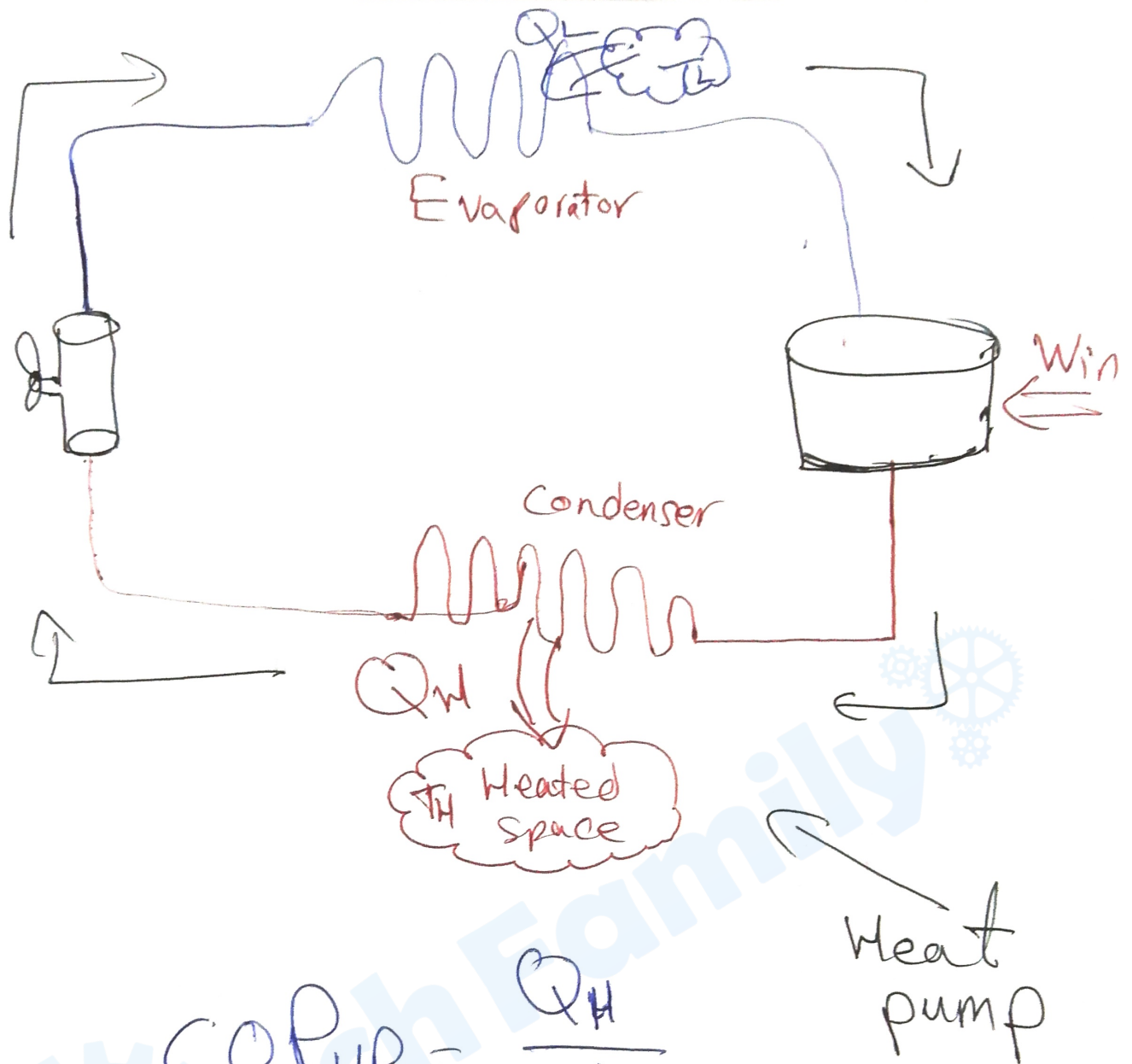
$$(Q_{in} + W_{in}) - (W_{out} + Q_{out}) = 0$$

$$Q_L + W_{in} - Q_H = 0$$

$$W_{in} = Q_H - Q_L$$

* The efficiency of the cycle is expressed by coefficient of performance

$$COP_R = \frac{Q_L}{W_{in}}$$



$$COP_{HP} = \frac{Q_H}{W_{in}}$$

* خفزة عمل الـ HP والـ R ، انه
يتم فيه حراره من المكان الابرد الى
المكان الاسخن بطريقة منطقيه
ولا تخالف قوانين الـ Thermodynamics
و باستهلاك شغل

* Pumping Heat from low temperature medium (T_L) To high temperature medium (T_H)

$$COP_{HP} = COP_R + 1$$

$$COP_{HP} > COP_R$$

Reversible and irreversibilities:-

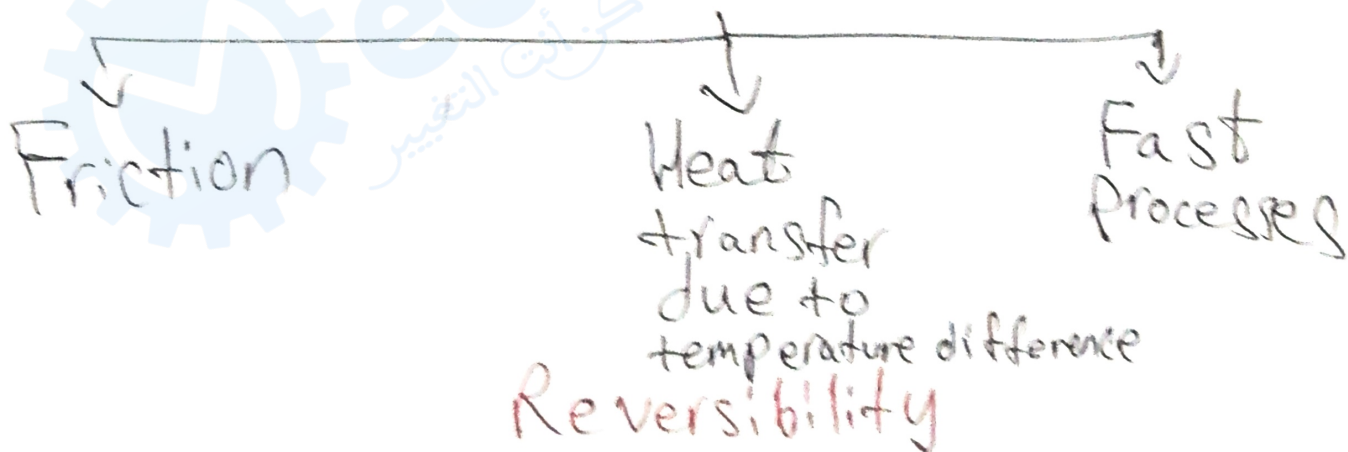
العمليات العكسية

Reversible process:-

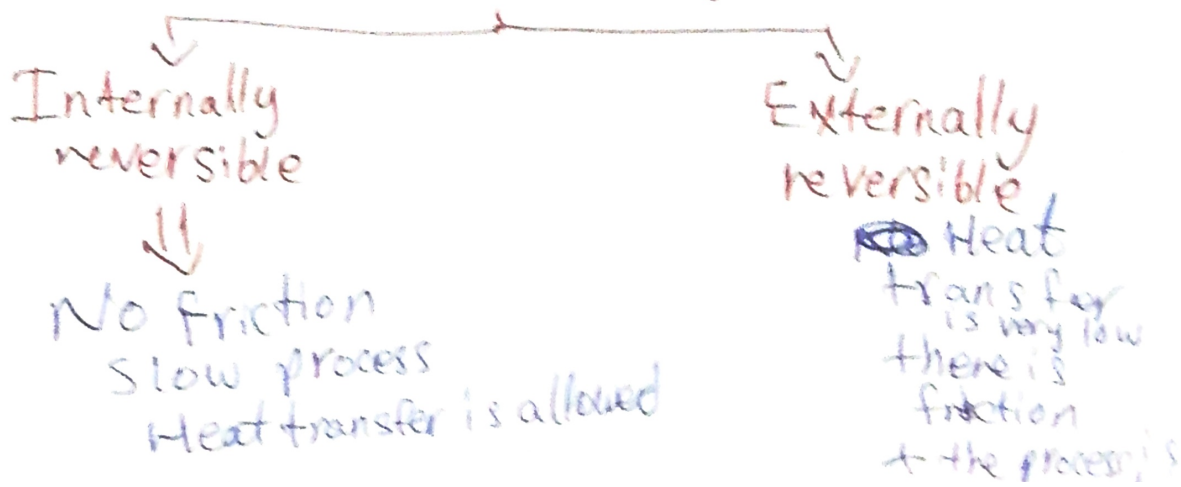
It's a process that doesn't contain any irreversibilities

العمليات العكسية

Irreversibilities



Reversibility



$$COP_{HP} = COP_R + 1$$

$$COP_{HP} > COP_R$$

Reversible and irreversibilities:—

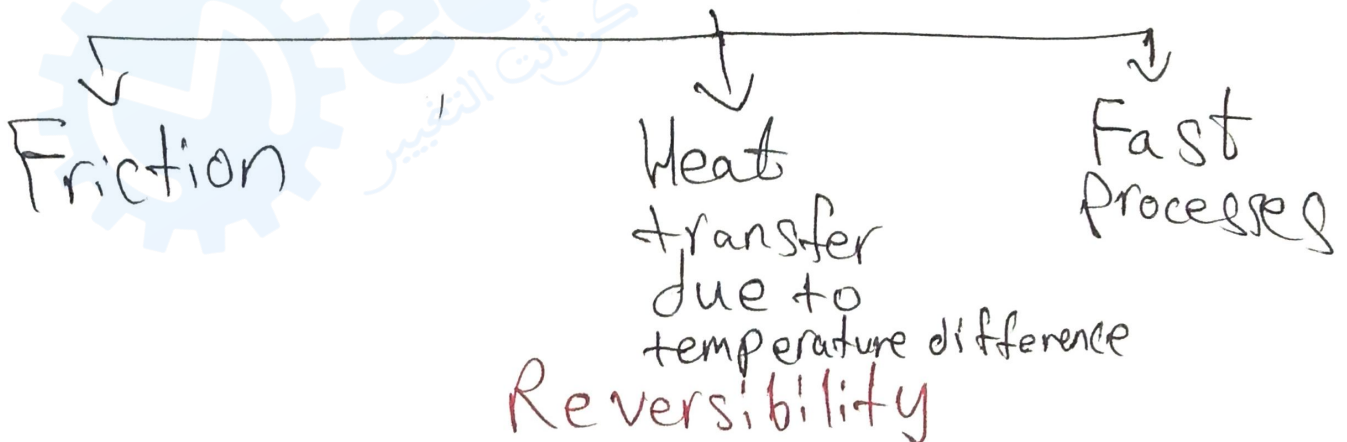
العمليات العكسية

Reversible process:—

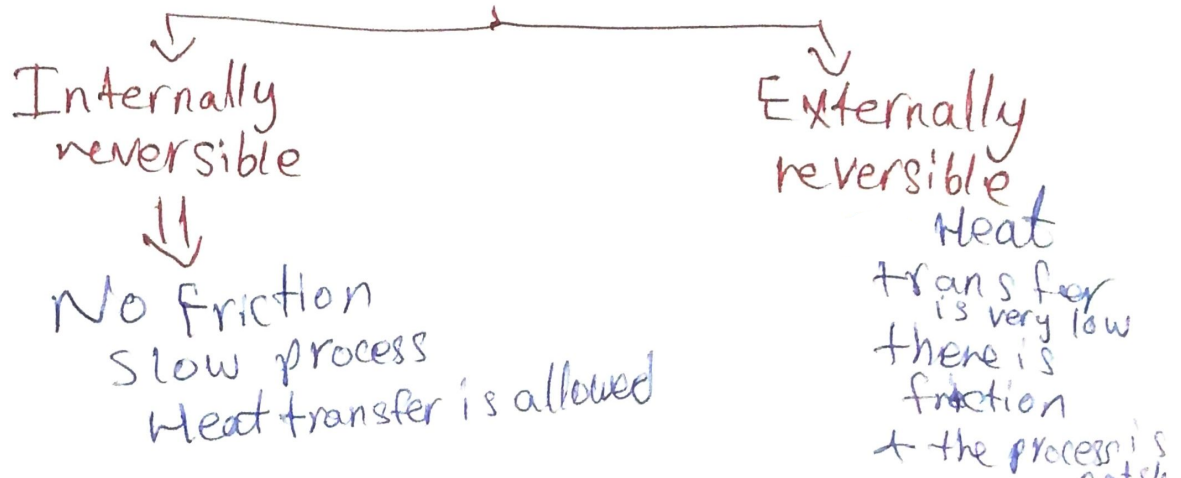
It's a process that doesn't contain any irreversibilities

لا يوجد فيها عكاسات

Irreversibilities



Reversibility



Externally + Internally reversible $\equiv$ Totally reversible

* كلما كانت irreversibilities اقل
كلما زادت كفاءة المحرك

Carnot cycle :-

هي دورة مثالية وخالية تماماً من
الخسائر
(Totally reversible)

Carnot heat engine :-

المحرك المثالي والخالي عن الخسائر
Heat engine

- No friction
- No temperature difference
(it's so low)
- slow processes

Carnot heat engine processes :-

- 1-2 reversible isothermal
heat addition & expansion)
- 2-3 reversible adiabatic
expansion
- 3-4 reversible isothermal
heat rejection (compression)
- 4-1 reversible adiabatic compression.

In carnot (reversible) :-

$$\left(\frac{Q_H}{Q_L} \right)_{\text{rev}} = \frac{T_H}{T_L}$$

$$\eta_{\text{th, carnot}} = 1 - \frac{T_L}{T_H}$$

T_L = Temperature of
heat sink in Kelvin

T_H = Temperature of heat
source.

* لا يوجد أي Heat engine كفاءة

تتعدى كفاءة η_{Carnot}

* كارنوت يعتبر عن أفضل أداء يمكن

أن يصله أي Heat engine

* لا يمكن تطبيق كارنوت في الواقع

* يعني لو لدينا مثلاً Steam power plant ، نستطيع

الاجزاء اعلى كفاءة يمكن ان تصلها

المحطة من خلال معرفة درجة

حرارة مصدر الحرارة (T_H) ودرجة حرارة

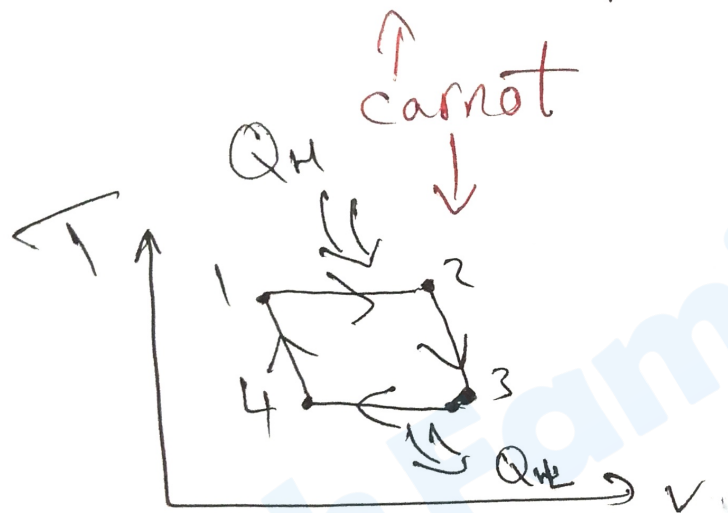
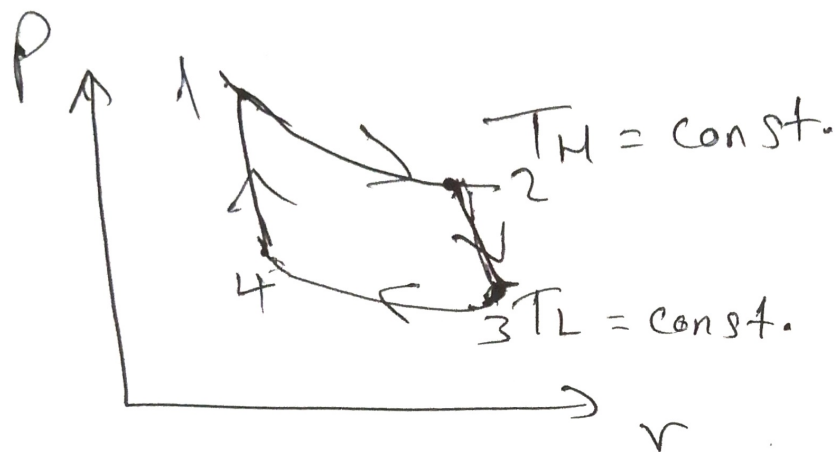
طارد الحرارة (المبرد) (T_L)

$$\eta_{\text{Carnot}} = 1 - \frac{T_L}{T_H}$$

لكن لا نستطيع ان نصل الى الاداء المثالي

(Carnot) ، لكن يمكننا محاولة الاقتراب

منه



* We have to know that, if we increase T_H (temp. of heat source), the η_{th} increases

Carnot refrigerator & Heat pump

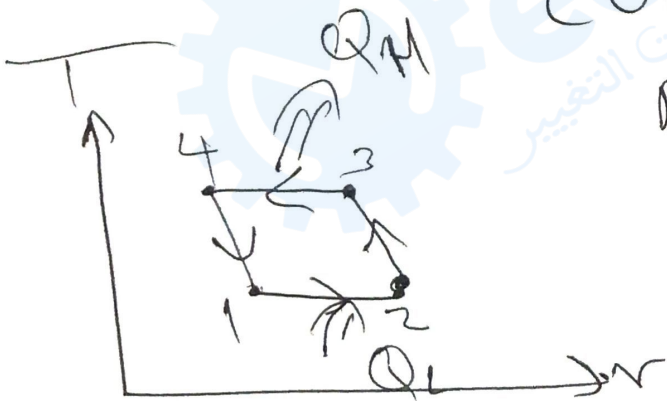
$$\text{COP}_R = \frac{Q_L}{W_{\text{in}}} = \frac{Q_L}{Q_H - Q_L}$$
$$= \frac{1}{Q_H/Q_L - 1}$$

$$\text{COP}_{R, \text{carnot}} = \frac{1}{T_H/T_L - 1}$$

$$\text{COP}_{\text{HP}, \text{carnot}} = \frac{1}{1 - T_L/T_H}$$

COP_{carnot} is the highest possible of any refrigeration cycle.

* It represents the best performance



End of chapter 6

CHAPTER 1

ENTROPY

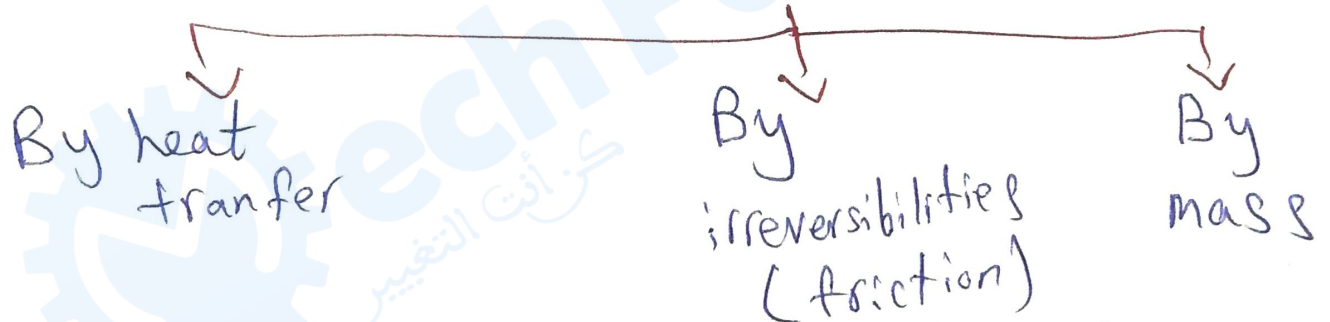
Entropy :-

It's a measure of molecular disorder or molecular randomness.

مقياس لدرجة الفوضى أو الاضطراب داخل النظام

* كلما كان ال entropy (الفوضى) اعلى ، كانت كفاءة الجهاز او العملية اقل ، فنحاول دائماً التقليل منه

How does entropy change (transfer)?



* The work doesn't produce entropy.

$$S_{in} - S_{out} + S_{gen} = \Delta S_{system}$$

Entropy balance equation

$$\Delta S_{\text{system}} = S_2 - S_1$$

Entropy
at
the
end of
the process
(final state)

Entropy
before
the
process
starts

$$S_{\text{in}} = \frac{Q_{\text{in}}}{T_H} + m S_{\text{in}}$$

Entropy
transferred
by heat

Entropy
transferred
by mass

$$S_{\text{out}} = \frac{Q_{\text{out}}}{T_L} + m S_{\text{out}}$$

$S_{\text{gen}} \equiv$ Entropy generated
by friction

Entropy calculations:-

1- For closed systems:-

$$\frac{Q_{in}}{T_H} - \frac{Q_{out}}{T_L} + S_{gen} = \Delta S_{system}$$

$$\Delta S_{system} = S_2 - S_1$$

* S at any state for any pure substance (R-134a/water) can be found from their tables.

$S_2 - S_1$ For ideal gases:-

$$\Delta S_{system} = C_v \ln \frac{T_2}{T_1} + R \ln \frac{V_2}{V_1}$$

OR

$$\Delta S_{system} = C_p \ln \frac{T_2}{T_1} - R \ln \frac{P_2}{P_1}$$

For constant specific heats

$$\Delta S_{\text{system}} = S_2^{\circ} - S_1^{\circ} - R \ln \frac{P_2}{P_1}$$

for variable specific heat

S° can be found from ideal gas tables (A-17 - - -)

$$* S^{\circ}(T)$$

For steady flow systems,

$$\dot{S}_{\text{in}} - \dot{S}_{\text{out}} + \dot{S}_{\text{gen}} = dS/dt$$

In steady flow systems, there is no change in entropy.

$$\left(\frac{\dot{Q}_{\text{in}}}{T_H} + \dot{m} S_{\text{in}} \right) - \left(\frac{\dot{Q}_{\text{out}}}{T_L} + \dot{m} S_{\text{out}} \right) + \dot{S}_{\text{gen}} = 0$$

$$\frac{\dot{Q}_{\text{in}}}{T_H} - \frac{\dot{Q}_{\text{out}}}{T_L} + \dot{m} (S_{\text{in}} - S_{\text{out}}) + \dot{S}_{\text{gen}} = 0$$

* S_{in} & S_{out} for pure substances can be found from their tables.

For ideal gases

$$\left[\begin{array}{l} S_{in} - S_{out} = S_{in}^{\circ} - S_{out}^{\circ} - R \ln \frac{P_{in}}{P_{out}} \\ \text{variable specific} \\ \text{heats} \end{array} \right.$$

$$S_{in} - S_{out} = C_v \ln \frac{T_{in}}{T_{out}} + R \ln \frac{V_{in}}{V_{out}}$$

I sentropic process :-

It's a constant entropy process.

$$S_1 = S_2$$

If we have a reversible adiabatic process, then we have to know that it's an isentropic process

$$\text{reversible} \Rightarrow S_{gen} = 0$$

$$\text{adiabatic} \Rightarrow S_{heat} = 0$$

$$\text{isentropic} \Rightarrow \Delta S_{system} = 0$$

Minimizing compressor's work:-

$$W_{rev, in} = \int v dp \quad (\text{Area under } v-p \text{ diagram})$$

I sentropic :- $(Pv^k = C)$

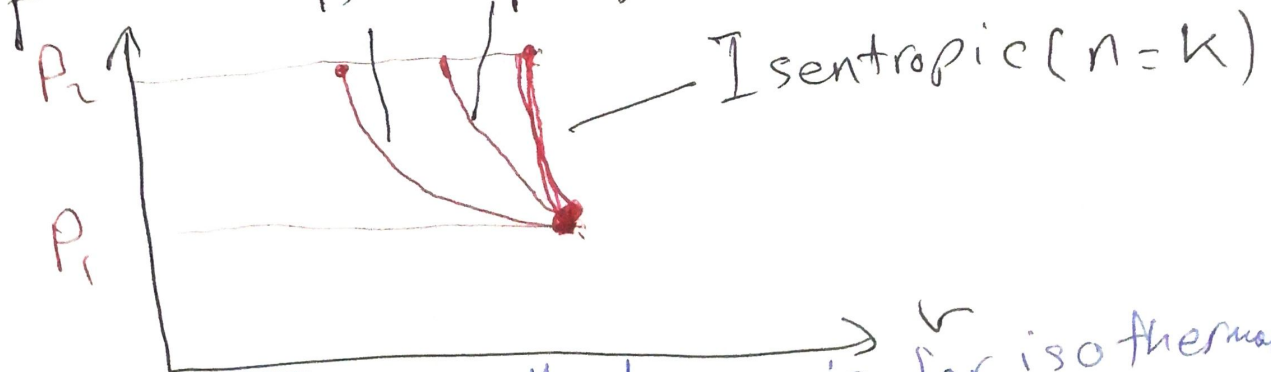
$$W_{comp, in} = \frac{kR(T_2 - T_1)}{k-1} = \frac{kRT_1}{k-1} \left[\left(\frac{P_2}{P_1} \right)^{(k-1)/k} - 1 \right]$$

Polytropic :- $(Pv^n = C)$

$$W_{comp, in} = \frac{nR(T_2 - T_1)}{n-1} = \frac{nRT_1}{n-1} \left[\left(\frac{P_2}{P_1} \right)^{n-1/n} - 1 \right]$$

Isothermal :- $(Pv = C)$

$$W_{comp, in} = RT \ln \frac{P_2}{P_1}$$



The smallest area is for isothermal, so W_{in} of isothermal is the minimum one

I sentropic efficiencies of steady flow systems:-

تعبير عن نسبة الا استفادة من الحالة
المثالية

$$\eta_{\text{Turbine, is.}} = \frac{\text{Actual turbine work}}{\text{Isentropic turbine work}}$$
$$= \frac{h_1 - h_{2a}}{h_1 - h_{2s}}$$

$\downarrow$
 $s_{1a} = s_{2a}$
 $s_{1s} = s_{2s}$

$$\eta_{\text{compressor, is}} = \frac{\text{Isentropic compressor work}}{\text{Actual compressor work}}$$
$$= \frac{h_{2s} - h_1}{h_{2a} - h_1}$$

$$\eta_{\text{Nozzle, is}} = \frac{\text{Actual K.E. at the exit}}{\text{Isentropic K.E. at the exit}}$$
$$= \frac{V_{2a}^2}{V_{2s}^2}$$

$$h_1 = h_{2a} + \frac{V_{2a}^2}{2}$$
$$h_1 = h_{2s} + \frac{V_{2s}^2}{2}$$

End of chapter 7