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الاسم: (جعفر العبدالله) الرقم الجامعي: 0110524 التسلسلي: 15

	Statement	
1	The most suitable solution for the year round heating and cooling of a hotel room is a split unit.	F
2	The outside air design temperature for calculating the heating load for a given application is selected to be the minimum outdoor temperature of the total hours during winter season.	F
3	The wind chill temperature expresses the combined effect of outdoor temperature and wind speed on human comfort.	T
4	Underfloor heating system is characterized by its high cost and fast dynamic response.	T A
5	The higher the depth of the wall below grade, the higher its overall heat transfer coefficient.	F
6	Fan coil units are direct expansion dx systems.	F
7	If the return air is treated to the reach the supply conditions without entering fresh air, then the cooling coil and the cooling load of the building are the same.	T
8	The latent load due to occupants can be detected by using temperature sensors.	F
9	When the occupant number is large, such as theater, then the CLF <sub>occ</sub> . Is taken as 1.	T
10	In the calculation of the heating load, the heat gains due to occupants, lights, appliances and solar effect are subtracted from the calculated heating load.	F



الاسم: (جعفر عاصم الحسيني) الرقم التسلسلي: ٠١١٥٧٤

**Question 1. [6 points]**

Fifty employees at a telecommunications center are working in an open space, The space is maintained at 20 °C dry bulb temperature and 50% RH. The office is open from 8:00 am till 1:00 pm when employees leave for a lunch break. Calculate the following:

- Total heat gain rate due to occupants at 10:00 am.
- Total and latent heating loads due to ventilation if the outside fresh air is available at 5 °C dry bulb temperature and 60% RH.

**Question 2. [4 points]** A solid wall consists of firm stone of 30 mm thickness, concrete of 100 mm thickness ( $\rho = 2300 \text{ kg/m}^3$ ), polystyrene layer of unknown thickness, cement bricks with air gaps of 70 mm thickness ( $k = 0.90 \text{ W/m.K}$ ), cement plaster of 20 mm thickness ( $k = 1.25 \text{ W/m.K}$ ). If the wall is to satisfy the Jordanian code with an overall heat transfer coefficient of 0.57 W/m<sup>2</sup>.K, how much should be the insulation thickness?

**Question 3. [3 points]**

A Jordanian family of six members with low hot water consumption at 50 °C, assume that the supply cold water temperature is 15 °C, and the boiler is required to heat the water in two hours, the boiler capacity is?

**Question 4. [7 points]**

Outside air at 38 °C dry bulb temperature and 60% relative humidity enters a cooling and dehumidifying device at a volumetric flow rate of 3800 m<sup>3</sup>/h. The air leaves the device at 90% relative humidity. The inside design condition is 26 °C and 45% relative humidity. The sensible heat ratio is 0.75 (assume 100% fresh air)

- Sketch the process on the psychrometric chart, and calculate the following
- Apparatus dew point.
- Dry bulb and wet bulb temperatures of the supply air.
- Cooling capacity of the air conditioner.
- Amount of moisture removed.
- Mass of chilled water for the cooling coil, if the supply and return temperatures from the chiller are 7°C / 12°C respectively



اسم الطالب لؤي عصام الخصين رقمه الجامعي 5110524

الكلية

المستوى

المادة تكييف ١

القسم

$$20 + 4.5 = \boxed{24.5}$$

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التاريخ

Q1) 50 employees (offices <sup>ات</sup><sub>open space</sub>) ,  $T_i = 20^\circ C$  ,  $RH = 50\%$  , 8 am - 1 pm

$$\begin{aligned} a) \dot{Q}_{s+D} \dot{Q}_{occ} &= n ( \dot{q}_{sen} * CLF + \dot{q}_{lat} ) \\ &= 50 ( 70 * 0.595 + 45 ) \\ &= 4332.5 \text{ W} \end{aligned}$$

$$b) \dot{V} = 10 * 50 = 500 \text{ L/s} = 0.5 \text{ m}^3/\text{s}$$

$$\dot{V}_0 = 0.79 \text{ m}^3/\text{kg} \text{ dry air}$$

$$\dot{m}_{0,A} = \frac{0.5}{0.79} = 0.633 \text{ kg/s}$$

$$\begin{aligned} \dot{Q}_{tot} &= \dot{m}_{0,A} \Delta h , h_i = 38 \text{ kJ/kg} \\ &= 0.633 (38 - 13) \quad h_o = 13 \text{ kJ/kg} \\ &= 15.825 \text{ kW} \end{aligned}$$

$$\begin{aligned} \dot{Q}_s &= \dot{m}_{0,A} C_p \Delta T \\ &= (0.633)(1)(20 - 5) = 9.495 \text{ kW} \end{aligned}$$

$$\dot{Q}_{lat} = 6.33 \text{ kW} \quad (\dot{Q}_{tot} - \dot{Q}_s)$$

Q2) Firm stone ;  $x = 0.03 \text{ m}$ ,  $K = 1.7$   
 concrete ;  $x = 0.1 \text{ m}$ ,  $K = 1.75$   $\text{W/m}\cdot\text{K}$ ,  
 polystyrene ;  $x = ?$ ,  $K = 0.03$   
 cement brick with AG ;  $x = 0.02 \text{ m}$ ,  $K = 0.9$   
 cement plaster ;  $x = 0.02 \text{ m}$ ,  $K = 1.25$

~~Overall~~  $U_w = 0.57 \text{ W/m}^2\cdot\text{K}$

$$U_w = \frac{1}{R_i + R_o + R_{fs} + R_c + R_p + R_{cb} + R_{cp}}$$

(Assume to obtain  $R_i, R_o$ )

$$0.57 = \frac{1}{0.12 + 0.06 + \frac{0.03}{1.7} + \frac{0.1}{1.75} + \frac{x}{0.03} + \frac{0.07}{0.9} + \frac{0.02}{1.25}}$$

~~MECH Family~~  $x_{pol} = 0.0422 \text{ m}$

Q3) family  $n=6$ , low hot water consum. (a)  $50^\circ\text{C}$

$T_{sup} = 15^\circ$ , Time to heat = 2 hours

$$H_w/\text{day} = (50)(2) + 4(30) = 220 \text{ L/day}$$

$$Q_{boiler} = \dot{m}_w C_w \Delta T$$

$$= \frac{220}{2 \times 60 \times 60} (41186) (50 - 15) = 4.48 \text{ kW}$$

②

(Q4) O.A, M

$$T_d = 38^\circ C$$

$$RH = 60\%$$

$$\dot{V} = 3800 \text{ m}^3/\text{h} = 1.056 \text{ m}^3/\text{s}$$

$$h = 104 \text{ kJ/kg}$$

$$w = 0.0256 \text{ kg}_{H_2O}/\text{kg}$$

$$A = 0.92 \text{ m}^2/\text{kg}_{H_2O}$$

$$\dot{m}_s = 1.148 \text{ kg/s}$$

S

$$RH = 90\%$$

$$SHR = 0.75$$

$$T_{db} = 11^\circ C$$

$$h = 30 \text{ kJ/kg}$$

$$w = 0.0074$$

$$T_{wb} = 10^\circ C$$

i, ~~h~~

$$T_d = 26$$

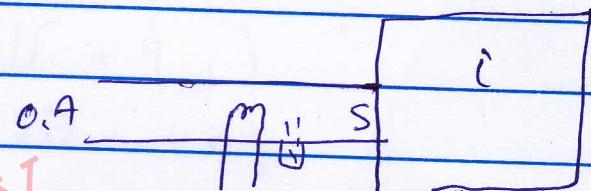
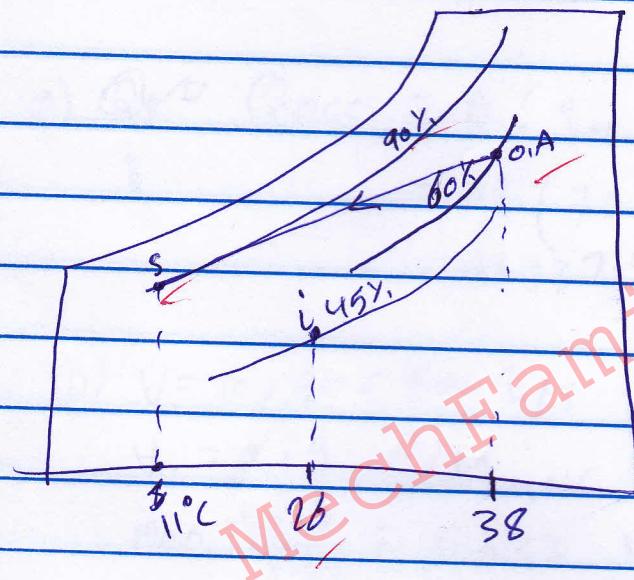
$$RH = 45\%$$

$$w = 0.0097 \text{ kg}_{H_2O}/\text{kg}$$

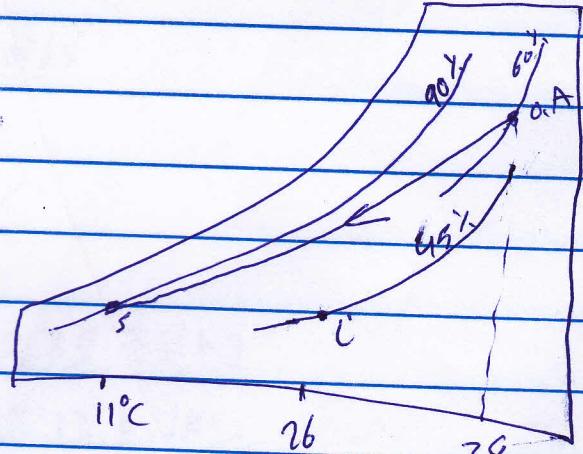
$$h = 50.5 \text{ kJ/kg}$$

$$i = 104 \text{ kJ/kg}$$

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$$\text{dew point} = 13^\circ C$$



$$\begin{aligned} Q_{cool} &= \dot{m}_s \Delta h = \dot{m}_s (h_{O.A} - h_s) \\ &= (1.148) (104 - 30) = 84.939 \text{ kW} \end{aligned}$$

$$\dot{m}_{cool} = \dot{m}_s (w_{O.A} - w_s)$$

$$= (1.148) (0.0256 - 0.0074) = 0.02089 \text{ kg}_{H_2O}$$

$$\dot{m}_{CH} = \frac{Q_{cool}}{c_p \Delta T} = \frac{84.939}{(4.186)(12-7)} = 4.058 \text{ kg/s}$$